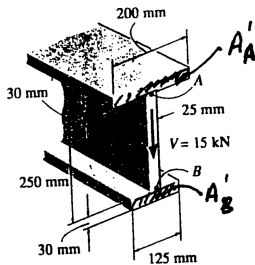


7-1 If the beam is subjected to a shear of $V = 15 \text{ kN}$, determine the web's shear stress at A and B . Indicate the shear-stress components on a volume element located at these points. Set $w = 125 \text{ mm}$. Show that the neutral axis is located at $\bar{y} = 0.1747 \text{ m}$ from the bottom and $I_{NA} = 0.2182(10^{-3}) \text{ m}^4$.



$$\bar{y} = \frac{(0.015)(0.125)(0.03) + (0.155)(0.025)(0.25) + (0.295)(0.2)(0.03)}{0.125(0.03) + (0.025)(0.25) + (0.2)(0.03)} = 0.1747 \text{ m}$$

$$I = \frac{1}{12}(0.125)(0.03^3) + 0.125(0.03)(0.1747 - 0.015)^2$$

$$+ \frac{1}{12}(0.025)(0.25^3) + 0.25(0.025)(0.1747 - 0.155)^2$$

$$+ \frac{1}{12}(0.2)(0.03^3) + 0.2(0.03)(0.295 - 0.1747)^2 = 0.218182(10^{-3}) \text{ m}^4$$

$$Q_A = \bar{y}A'_A = (0.310 - 0.015 - 0.1747)(0.2)(0.03) = 0.7219(10^{-3}) \text{ m}^3$$

$$Q_B = \bar{y}A'_B = (0.1747 - 0.015)(0.125)(0.03) = 0.59883(10^{-3}) \text{ m}^3$$

$$\tau_A = \frac{VQ_A}{I t} = \frac{15(10^3)(0.7219)(10^{-3})}{0.218182(10^{-3})(0.025)} = 1.99 \text{ MPa} \quad \text{Ans}$$

$$\tau_B = \frac{VQ_B}{I t} = \frac{15(10^3)(0.59883)(10^{-3})}{0.218182(10^{-3})(0.025)} = 1.65 \text{ MPa} \quad \text{Ans}$$

$$\tau_A = 1.99 \text{ MPa}$$



$$\tau_B = 1.65 \text{ MPa}$$

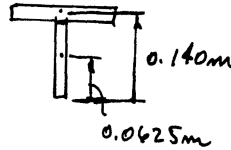
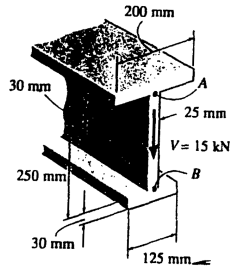
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7-2 If the wide-flange beam is subjected to a shear of $V = 30 \text{ kN}$, determine the maximum shear stress in the beam. Set $w = 200 \text{ mm}$.



Section Properties :

$$I = \frac{1}{12}(0.2)(0.310)^3 - \frac{1}{12}(0.175)(0.250)^3 = 268.652(10)^{-6} \text{ m}^4$$

$$Q_{\max} = \Sigma \bar{y}A = 0.0625(0.125)(0.025) + 0.140(0.2)(0.030) = 1.0353(10)^{-3} \text{ m}^3$$

$$\tau_{\max} = \frac{VQ}{It} = \frac{30(10)^3 (1.0353)(10)^{-3}}{268.652(10)^{-6} (0.025)} = 4.62 \text{ MPa} \quad \text{Ans}$$

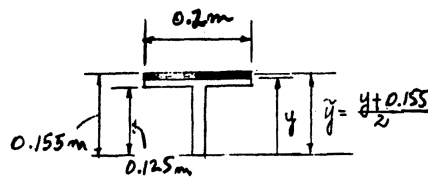
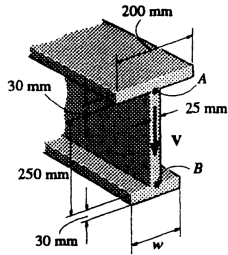
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7-3 If the wide-flange beam is subjected to a shear of $V = 30 \text{ kN}$, determine the shear force resisted by the web of the beam. Set $w = 200 \text{ mm}$.



$$I = \frac{1}{12}(0.2)(0.310)^3 - \frac{1}{12}(0.175)(0.250)^3 = 268.652(10)^{-6} \text{ m}^4$$

$$Q = \left(\frac{0.155 + y}{2}\right)(0.155 - y)(0.2) = 0.1(0.024025 - y^2)$$

$$\tau_f = \frac{30(10)^3(0.1)(0.024025 - y^2)}{268.652(10)^{-6}(0.2)}$$

$$\begin{aligned} V_f &= \int \tau_f dA = 55.8343(10)^6 \int_{0.125}^{0.155} (0.024025 - y^2)(0.2) dy \\ &= 11.1669(10)^6 \left[0.024025y - \frac{1}{3}y^3 \right]_{0.125}^{0.155} \end{aligned}$$

$$V_f = 1.457 \text{ kN}$$

$$V_w = 30 - 2(1.457) = 27.1 \text{ kN} \quad \text{Ans}$$

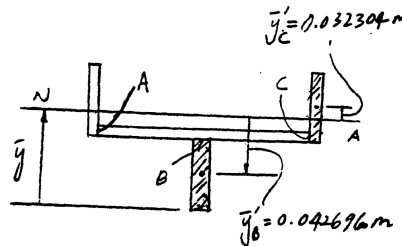
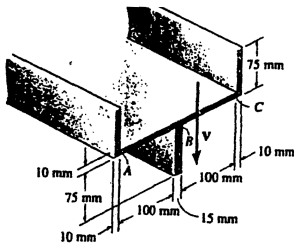
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*7-4. The beam is fabricated from three steel plates, and it is subjected to a shear force of $V = 150$ kN. Determine the shear stress at points A and C where the plates are joined. Show $\bar{y} = 0.080196$ m from the bottom and $I_{NA} = 4.8646(10^{-6})$ m⁴.



$$\bar{y} = \frac{\sum \bar{y}A}{\sum A} = \frac{0.0375(0.075)(0.015) + 0.08(0.215)(0.01) + 2[0.1125(0.075)(0.01)]}{0.075(0.015) + 0.215(0.01) + 2(0.075)(0.01)} = 0.080196 \text{ m}$$

$$I = \frac{1}{12}(0.015)(0.075^3) + (0.015)(0.075)(0.080196 - 0.0375)^2 + \frac{1}{12}(0.215)(0.01^3) + 0.215(0.01)(0.080196 - 0.08)^2 + 2\left[\frac{1}{12}(0.01)(0.075^3) + 0.01(0.075)(0.1125 - 0.080196)^2\right] = 4.8646(10^{-6}) \text{ m}^4$$

$$Q_A = Q_C = \bar{y}'_C A = 0.032304(0.075)(0.01) = 24.2277(10^{-6}) \text{ m}^3$$

$$\tau_A = \tau_C = \frac{VQ}{It} = \frac{150(10^3)(24.2277)(10^{-6})}{4.8646(10^{-6})(0.01)} = 74.7 \text{ MPa} \quad \text{Ans}$$

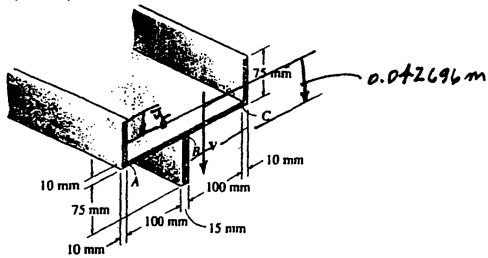
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7-5. The beam is fabricated from three steel plates, and it is subjected to a shear force of $V = 150$ kN. Determine the shear stress at point B where the plates are joined. Show $\bar{y} = 0.080196$ m from the bottom and $I_{NA} = 4.8646(10^{-6})$ m⁴.



$$\bar{y} = \frac{\sum \bar{y}A}{\Sigma A} = \frac{0.0375(0.075)(0.015) + 0.08(0.215)(0.01) + 2[0.1125(0.075)(0.01)]}{0.075(0.015) + 0.215(0.01) + 2(0.075)(0.01)} = 0.080196 \text{ m}$$

$$I = \frac{1}{12}(0.015)(0.075^3) + (0.015)(0.075)(0.080196 - 0.0375)^2 + \frac{1}{12}(0.215)(0.01^3) + 0.215(0.01)(0.080196 - 0.08)^2 + 2\left[\frac{1}{12}(0.01)(0.075^3) + 0.01(0.075)(0.1125 - 0.080196)^2\right] = 4.8646(10^{-6}) \text{ m}^4$$

$$Q_B = \bar{y}_B A = 0.042696(0.075)(0.015) = 48.0333(10^{-6}) \text{ m}^3$$

$$\tau_B = \frac{VQ_B}{I t} = \frac{150(10^3)(48.0333)(10^{-6})}{4.8646(10^{-6})(0.015)} = 98.7 \text{ MPa} \quad \text{Ans}$$

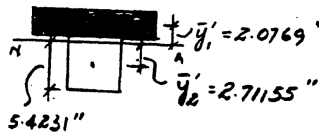
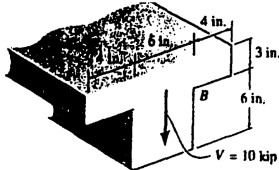
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7-6. If the T-beam is subjected to a vertical shear of $V = 10$ kip, determine the maximum shear stress in the beam. Also, compute the shear-stress jump at the flange-web junction AB . Sketch the variation of the shear-stress intensity over the entire cross section. Show that $I_{NA} = 532.04 \text{ in}^4$.



$$\bar{y} = \frac{(1.5)(3)(14) + 6(6)(6)}{3(14) + 6(6)} = 3.5769 \text{ in.}$$

$$I = \frac{1}{12}(14)(3)^3 + 3(14)(3.5769 - 1.5)^2 + \frac{1}{12}(6)(6)^3 + 6(6)(6 - 3.5769)^2 = 532.04 \text{ in}^4$$

$$Q_{\max} = \bar{y}_2 A' = 2.71155(5.4231)(6) = 88.23 \text{ in}^3$$

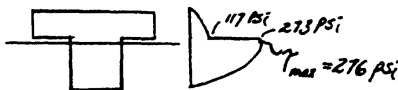
$$\tau_{\max} = \frac{VQ_{\max}}{I t} = \frac{10(10^3)(88.23)}{532.04(6)} = 276 \text{ psi} \quad \text{Ans}$$

$$Q_{AB} = \bar{y}_1 A' = 2.0769(3)(14) = 87.23 \text{ in}^3$$

$$(\tau_{AB})_f = \frac{VQ_{AB}}{I t_f} = \frac{10(10^3)(87.23)}{532.04(14)} = 117.1 \text{ psi}$$

$$(\tau_{AB})_w = \frac{VQ_{AB}}{I t_w} = \frac{10(10^3)(87.23)}{532.04(6)} = 273.3 \text{ psi}$$

$$\begin{aligned} \text{Shear stress jump} &= (\tau_{AB})_w - (\tau_{AB})_f \\ &= 273.3 - 117.1 \\ &= 156 \text{ psi} \quad \text{Ans} \end{aligned}$$



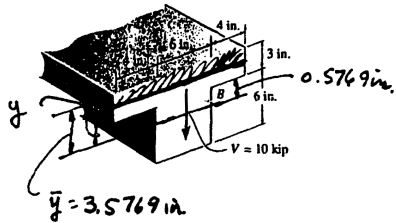
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7-7. If the T-beam is subjected to a vertical shear of $V = 10$ kip, determine the vertical shear force resisted by the flange. Show that $I_{NA} = 532.04 \text{ in}^4$.



$$\bar{y} = \frac{(1.5)(3)(14) + 6(6)(6)}{3(14) + 6(6)} = 3.5769 \text{ in.}$$

$$I = \frac{1}{12}(14)(3^3) + 3(14)(3.5769 - 1.5)^2 + \frac{1}{12}(6)(6^3) + 6(6)(6 - 3.5769)^2 = 532.04 \text{ in}^4$$

$$Q = (3.5769 - y)(14)\left(\frac{3.5769 + y}{2}\right) = 7(3.5769^2 - y^2)$$

$$\tau = \frac{10(7)(3.5769^2 - y^2)}{(532.04)(14)} = 0.009398(3.5769^2 - y^2)$$

$$\begin{aligned} V_f &= \int \tau dA = \int_{0.5769}^{3.5769} 0.009398(3.5769^2 - y^2)(14 dy) \\ &= 0.13157(3.5769^2 y - \frac{1}{3}y^3) \Big|_{0.5769}^{3.5769} \\ &= 3.05 \text{ kip} \quad \text{Ans} \end{aligned}$$

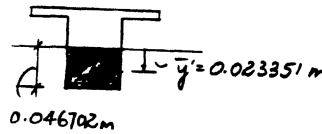
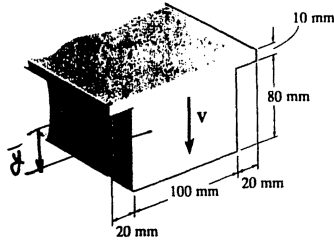
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*7-8 Determine the maximum shear stress in the strut if it is subjected to a shear force of $V = 15 \text{ kN}$. Show that $I_{NA} = 6.691(10^{-6}) \text{ m}^4$.



$$\bar{y} = \frac{(0.005)(0.01)(0.14) + (0.05)(0.1)(0.08)}{(0.01)(0.14) + (0.1)(0.08)} = 0.043298 \text{ m}$$

$$I = \frac{1}{12}(0.14)(0.01^3) + (0.14)(0.01)(0.043298 - 0.005)^2 + \frac{1}{12}(0.1)(0.08^3) + (0.1)(0.08)(0.05 - 0.43298)^2 = 6.6911(10^{-6}) \text{ m}^4$$

$$Q_{\max} = \bar{y}'A' = (0.023351)(0.046702)(0.1) = 0.1090544 (10^{-3}) \text{ m}^3$$

$$\tau_{\max} = \frac{VQ_{\max}}{It} = \frac{15(10^3)(0.1090544)(10^{-3})}{6.6911(10^{-6})(0.1)} = 2.44 \text{ MPa}$$

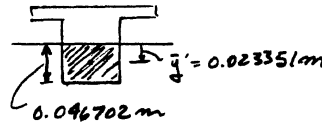
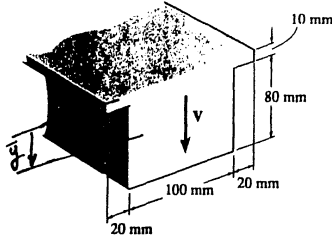
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7-9 Determine the maximum shear force V that the strut can support if the allowable shear stress for the material is $\tau_{\text{allow}} = 50 \text{ MPa}$. Show that $I_{NA} = 6.691(10^{-6}) \text{ m}^4$.



$$\bar{y} = \frac{(0.005)(0.01)(0.14) + (0.05)(0.1)(0.08)}{(0.01)(0.14) + (0.1)(0.08)} = 0.043298 \text{ m}$$

$$I = \frac{1}{12}(0.14)(0.01^3) + (0.14)(0.01)(0.043298 - 0.005)^2 + \frac{1}{12}(0.1)(0.08^3) + (0.1)(0.08)(0.043298 - 0.05)^2 = 6.6911(10^{-6}) \text{ m}^4$$

$$Q_{\text{max}} = \bar{y}'A' = (0.023351)(0.046702)(0.1) = 0.1090544 (10^{-3}) \text{ m}^3$$

$$\tau_{\text{max}} = \tau_{\text{allow}} = \frac{VQ_{\text{max}}}{It}$$

$$50(10^6) = \frac{V(0.1090544)(10^{-3})}{6.6911(10^{-6})(0.1)}$$

$$V = 307 \text{ kN} \quad \text{Ans}$$

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