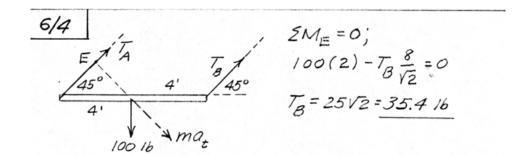
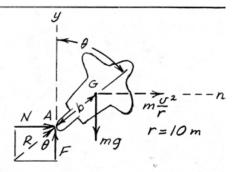


6/2 A 5m 5m 8  $8>A since <math>EM_C$  must be CCW. 3.5m G must be CCW.  $EM_D=0$ ; 5mg + 0.258(3.5) - 108 = 0 E=5mg/9.125 = 0.548mg EF=ma; 0.258=ma,  $a=\frac{0.25(0.548)mg}{m}$   $a=0.25(0.548)(9.81)=\frac{1.344}{m/5^2}$ 



6/5

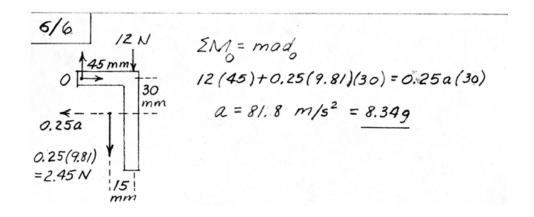


$$\sum_{A} = m\bar{a}d : mgb \sin\theta = m \frac{\sigma^2}{r} b \cos\theta, \quad \sigma^2 = gr \tan\theta$$
But  $\tan\theta = N/F = 1/\mu so \quad \sigma^2 = \frac{gr}{\mu}, \quad \sigma = \sqrt{\frac{9.81 \times 10}{0.70}}$ 

$$= 11.84 \text{ m/s}$$

$$\theta = tan^{-1} \frac{\sigma^2}{gr} = tan^{-1} \frac{11.84^2}{9.81 \times 10} = \frac{55.0^{\circ}}{9.81 \times 10^{\circ}}$$

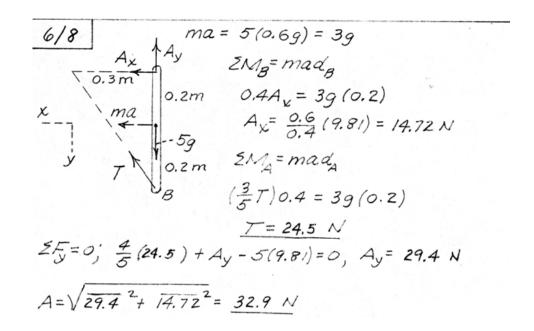
Note: The fact that in reality this is a rigid body rotating about the central axis does not invalidate the plane-motion analysis as a translating body so long as  $\theta = 0$ .

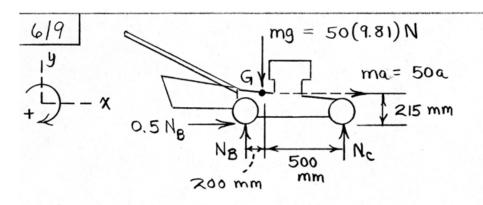


Tipping impends when  $N_A \rightarrow 0$ .

The ping impends when  $N_A \rightarrow 0$ .

The pi

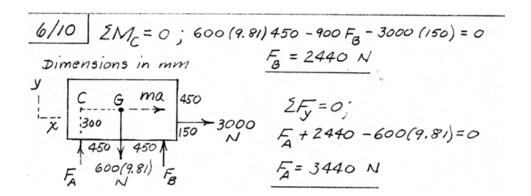


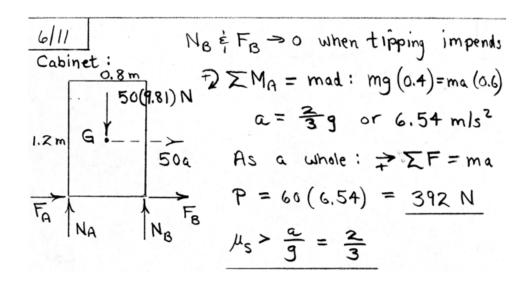


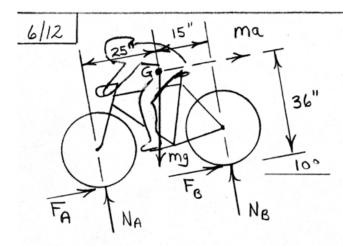
$$\Sigma Fy = 0$$
:  $N_B + N_c - 50 (9.81) = 0$ 

$$\Sigma Fy = 0$$
:  $N_B + N_C - 50 (9.81) = 0$   
 $\Sigma M_B = \text{mad}$ :  $50(9.81)(0.2) - N_C(0.7) = 50a(0.215)$ 

Simultaneous solution : 
$$\begin{cases} N_B = 414 N \\ N_C = 76.6 N \\ \alpha = 4.14 \text{ m/s}^2 \end{cases}$$

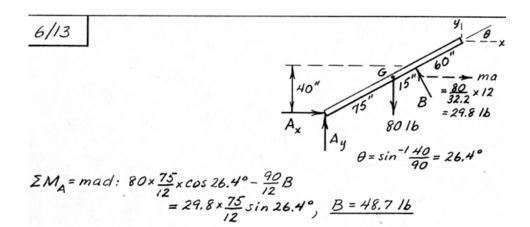


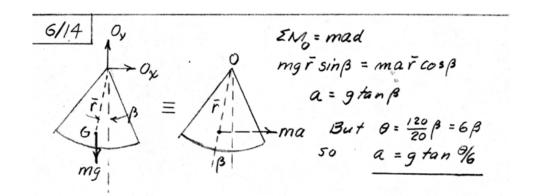


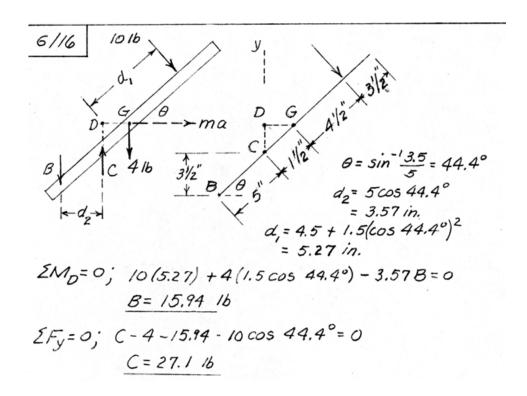


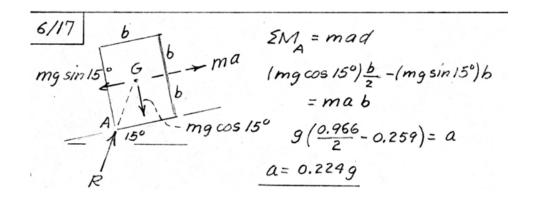
Tipping at front wheel: NB,  $F_B \rightarrow 0$ +2  $\sum M_A = \text{mad}$ : mg (25 cos 10° - 36 sin 10°) = ma (36)

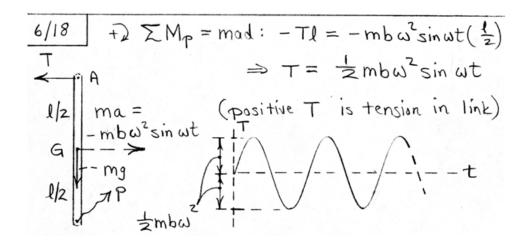
Solve to obtain a = 0.510g (16.43 ft/sec2)

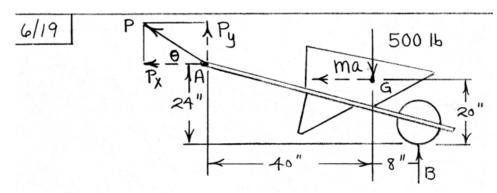








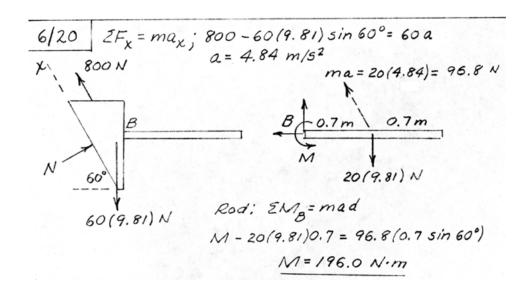


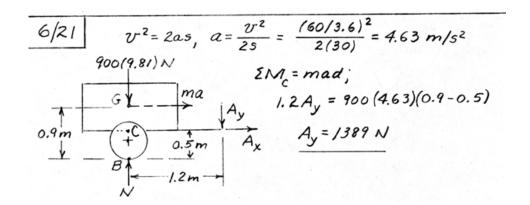


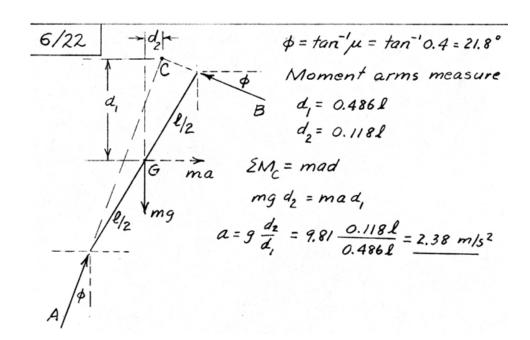
Static equilibrium : Px = ma = 0

$$\Re \sum M_{A} = 0$$
: 500(40) - B(48) = 0,  $\frac{B = 417 \text{ lb}}{\text{st}}$   
Dynamic:  $\Re \sum M_{A} = \text{mad}$ :

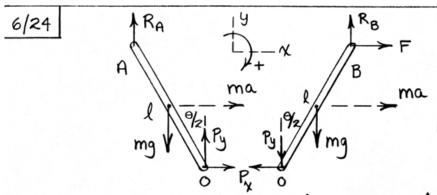
$$500(40) - B(48) = \frac{500}{32.2}(5)(4)$$
,  $B = 410 1b$   
 $+ \Sigma F_{\chi} = ma : P_{\chi} = \frac{500}{32.2}(5) = 77.6 lb$ .:  $P = 118.7 lb$   
 $+ \uparrow \Sigma F_{y} = 0 : B - 500 + P_{y} = 0$ ,  $P_{y} = 89.8 lb$ .  $\Theta = 49.2^{\circ}$ 







 $7 \sum_{mad} \sum_{mad} \left(\frac{1}{2} \sin \theta\right) = \sum_{mad} \left(\frac{1}{2} \cos \theta\right)$   $-mg \left(l \sin \theta\right) = \sum_{mad} \left(\frac{1}{2} \cos \theta\right)$   $+ mq \left(l \cos \theta\right)$  Simplify + o  $K\theta - \sum_{mad} mgl \sin \theta = \sum_{mad} mal \cos \theta$  V(2) With m = 0.5 kg, l = 0.6 m,  $a = 2g, and \theta = 200, K$ is found to be  $K = 46.8 \frac{N \cdot m}{rad}$ 



A ∑Mo= mad: Ralsin =-mg 是sin == ma 是cos =

B ZMo= mad: Flcos 皇 + mg 之 sin 是 - Rg l sin 皇 = ma 是 cos 皇

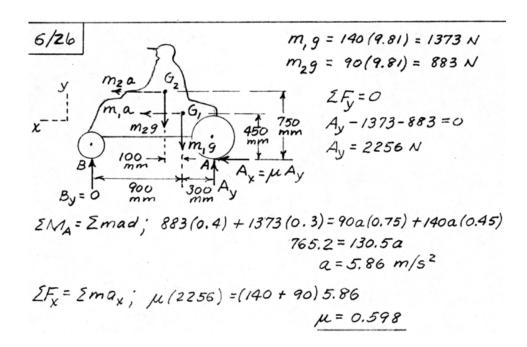
Two bars together:

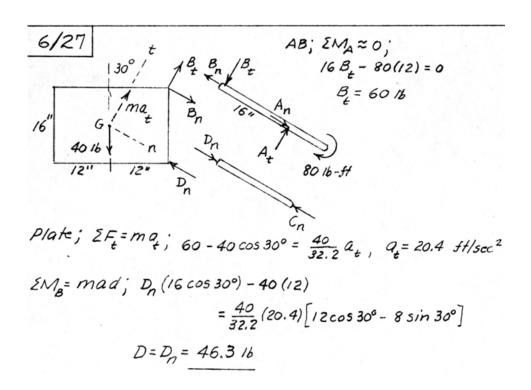
EFy=0: RA+R8-2mg=0

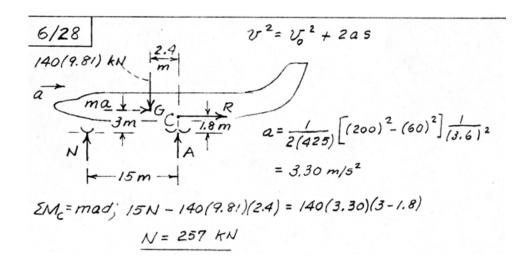
Subtract Eq. (A) from (B), combine with y-eq. to obtain  $\theta = 2 tan^{-1} \frac{F}{mg}$ 

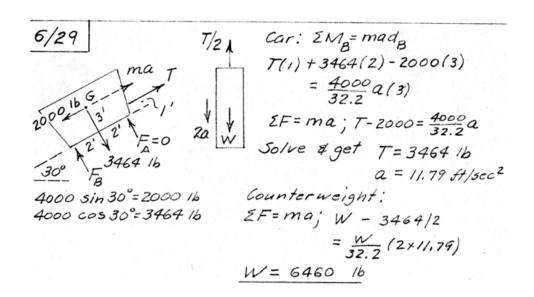
Both bars together:  $\sum F_{\chi} = ma_{\chi}$ :  $F = 2ma_{\chi}a = \frac{9}{2}tan\frac{9}{2}$ From B:  $mg tan \frac{9}{2} l cos \frac{9}{2} + mg \frac{1}{2} sin \frac{9}{2} - R_B l sin \frac{9}{2} = \frac{9}{2}tan \frac{9}{2} \frac{1}{2} cos \frac{9}{2}$ 

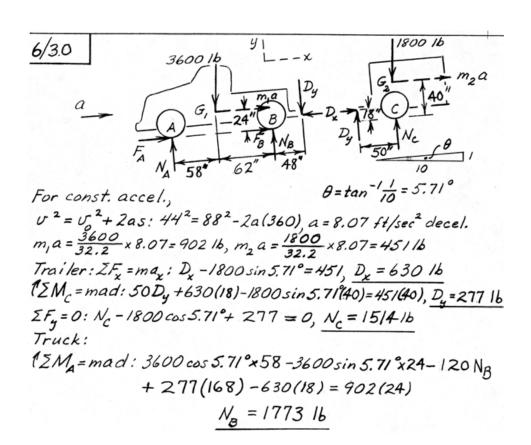
 $\Rightarrow \frac{R_8 = \frac{5}{4} \text{ mg}}{\text{Finally, from y-eq.}} \quad \frac{R_A = \frac{3}{4} \text{mg}}{R_A = \frac{3}{4} \text{mg}}$ 



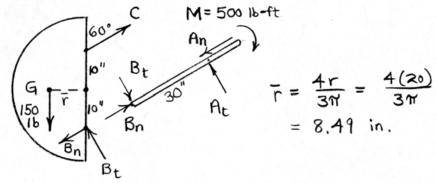




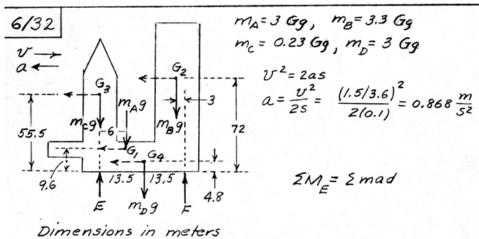




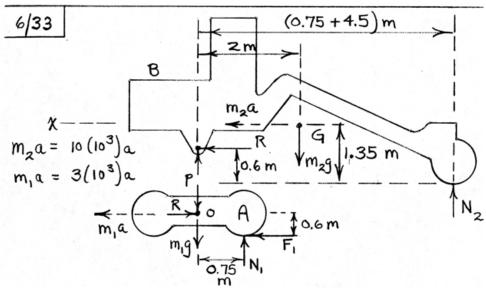
6/31 AB:  $\{ \sum M_A = 0 : 30 B_t = 500(12) \}_{0}^{1} B_t = 200 1b$   $\{ \sum F_t = 0 \Rightarrow A_t = 200 1b \}_{0}^{1}$ Plate:  $\{ \sum F_t = ma_t : 200 - 150 \frac{\sqrt{3}}{2} = \frac{150}{32.2} a_t \}_{0}^{1}$  $\{ a_t = 15.05 \text{ ft/sec}^2 \}_{0}^{2}$ 



$$\sum M_c = \text{mad}$$
:  $200(20)(\frac{1}{2}) + B_n(20\frac{\sqrt{3}}{2})$   
 $-150(8.49) = \frac{150}{32.2}15.05(\frac{\sqrt{3}}{2}8.49 + \frac{1}{2}10)$   
 $A_n = B_n = 8.03 \text{ lb}$ 



27F - [3(6) + 3.3(27-3) + 3(13.5)]9.81 = [3(9.6) + 3.3(72) + 0.23(55.5) + 3(4.8)]0.868 27F = [350.8 + 254.8, F = 59.5 MN

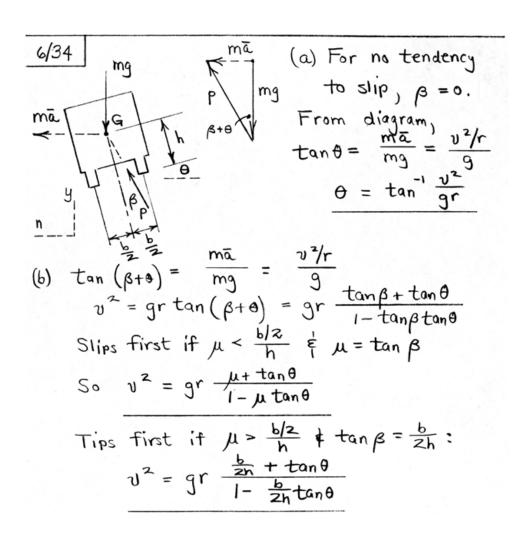


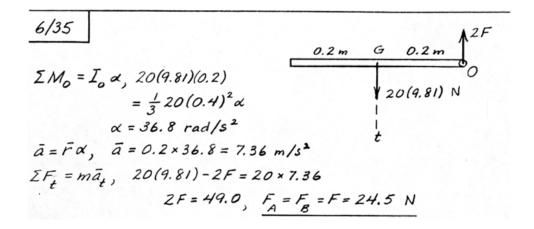
For rear wheels of unit A to lift off ground:

$$\bigoplus \sum_{N_1} m_1 \alpha d_1 : [P + 3(10^3)(9.81)](0.750) - 0.6R = 3(10^3)\alpha(0.6)$$

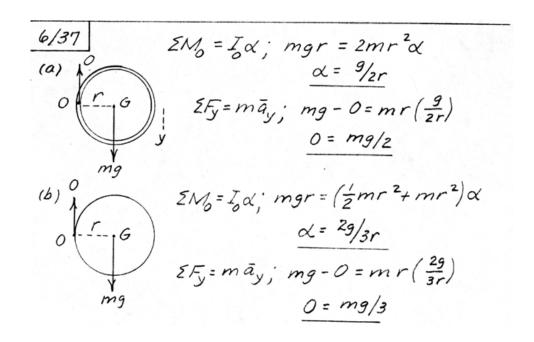
B 
$$\times M_{N_2} = m_2 a d_2$$
: 10 (10<sup>3</sup>)(9.81)(4.5 + 0.75 - 2)  
-P (4.5 + 0.75) + 0.6 R = 10 (10<sup>3</sup>) a (1.35)  
 $\times F_X = mq_X$ : R = 10 (10<sup>3</sup>) a

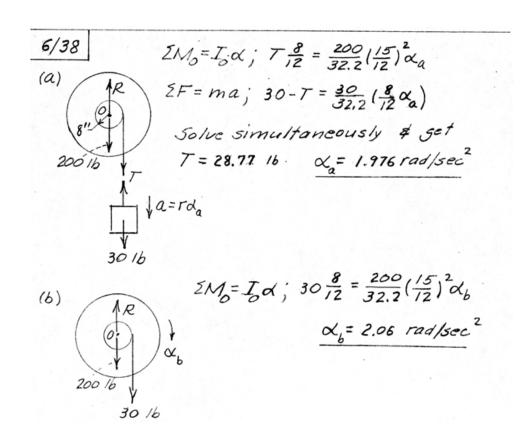
Solve the above three equations to obtain 
$$R = 76.2 \text{ kN}$$
,  $P = 49.8 \text{ kN}$ ,  $Q = 7.62 \text{ m/s}^2$   
For constant acceleration,  $S = \frac{v^2}{2a} = \frac{(40/3.6)^2}{2(7.62)} = \frac{8.10 \text{ m}}{2}$ 

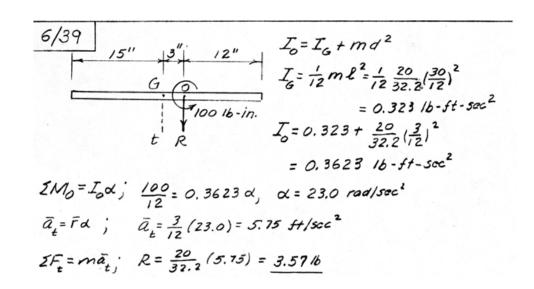




6/36 Accelerating force on rear wheels is  $F = ma = \frac{5200}{9} 0.59 = 2600 \, lb$   $\alpha_{drum} = \frac{q_t}{r} = \frac{0.5(32.2)}{3} = 5.37 \, rad/sec^2$   $EM_0 = I_0 \alpha; \quad I_0 = \frac{2600(3)}{5.37} = 1453 \, lb-ff-sec^2$ 







 $F \sum M_0 = I_0 \propto \text{ for drum}:$   $T_1(0.2) - T_2(0.3) - 2 = 8(0.225)^2 \propto (1)$   $+ \sum F = ma \text{ for } 12 - kg \text{ cylinder}:$   $12(9.81) - T_1 = 12(0.2\alpha) (2)$   $+1 \sum F = ma \text{ for } 7 - kg \text{ cylinder}:$   $T_2 \qquad T_2 - 7(9.81) = 7(0.3\alpha) (3)$  Solution of Eqs. (1) - (3):  $T_1 = 116.2 \text{ N}$   $T_2 = 70.0 \text{ N}$   $\alpha = 0.622 \text{ rod/s}^2$ 

For complete ring of mass 2m

moment of inertia about

diameter  $\chi - \chi = \frac{1}{2}(2mr^2)$ So moment of inertia  $\chi - \frac{1}{2}$   $\chi - \frac{1}{2}$ 

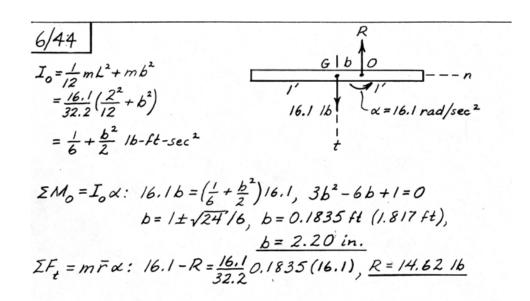
6/43

O 
$$\bar{I} = \frac{1}{12} m \ell^2 = \frac{1}{12} \frac{10}{32.2} (\frac{18}{12})^2 = 0.0582 \text{ H-16-sec}$$
 $q''' (Sol. I)$ 
 $2M_G = \bar{I}\alpha$ ;  $\frac{q}{12}P - 48\frac{q}{12} = 0.0582 \propto$ 
 $\ell''' = \frac{10}{10} lb$ 
 $2F_L = m\bar{\alpha}_L$ ;  $P + 48 = \frac{10}{32.2} \frac{q}{12} \propto$ 

Solve simultaneously & get

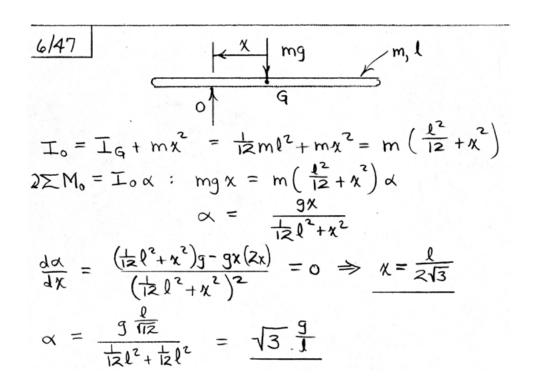
 $P = 96.0 \ lb$ ,  $\alpha = 618.2 \ rad/sec^2$ 

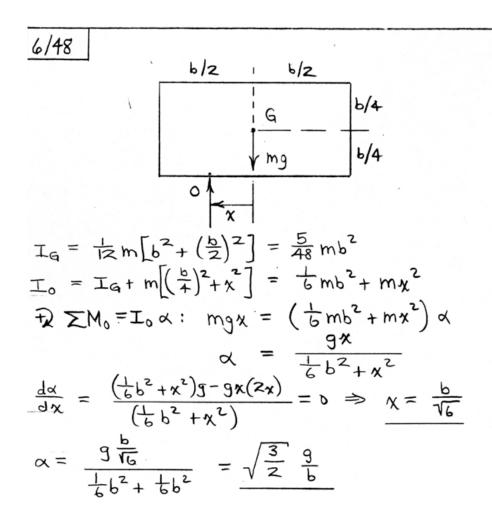
(Sol I)  $g = \frac{10}{12} l^2 = \frac{10}{32} l^2 = \frac{10}{322} l^2 = \frac{10}{322} l^2 = \frac{10}{3222} l^2 = \frac{$ 



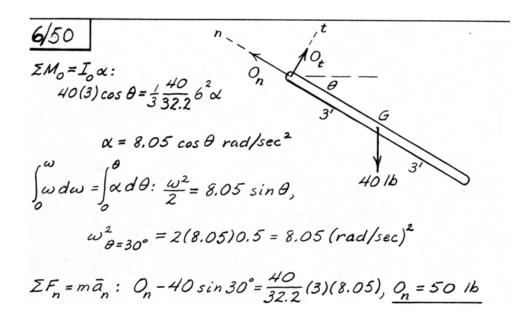
For slender rod,  $q = \frac{k_0^2}{r} = \frac{\frac{1}{3}L^2}{\frac{L}{2}} = \frac{2}{3}(6) = 4 \text{ ft}$ For fixed-axis rotation,  $ZM_Q = 0 \text{ at all times,}$ before, during, and after impact. R

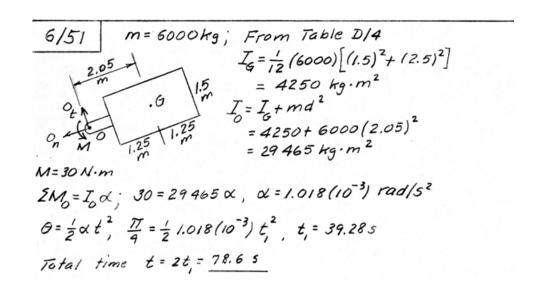
Thus  $40(1) \cos 30^\circ - 4 O_t = 0, O_t = 8.66 \text{ lb at all times}$ 

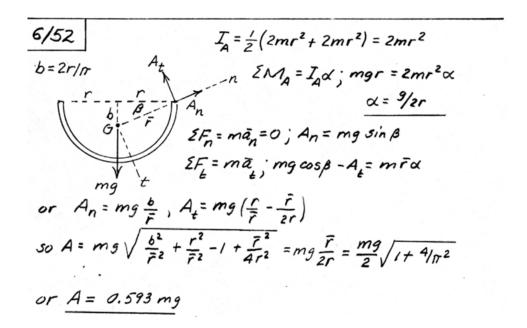




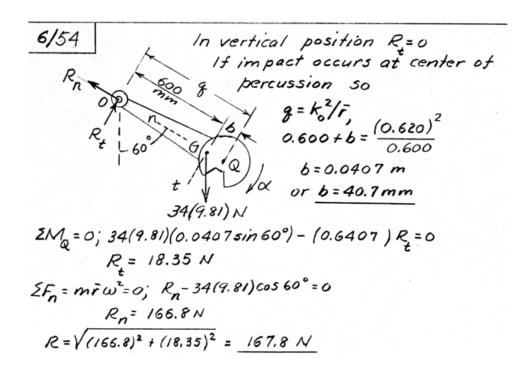
 $\frac{6/49}{\omega^{2} = \omega_{0}^{2} + 2\bar{\alpha}\theta, \left(\frac{1200 \times 2\pi}{60}\right)^{2} = 0 + 2\bar{\alpha}\left(18 \times 2\pi\right), \bar{\alpha} = 69.8 \text{ rad/s}^{2}}{60}$ Static test ZM = 0:  $0.660 - 2.8(9.81)\bar{r}$ ,  $\bar{r} = 0.0240 \text{ m}$ (a)  $ZM = I\alpha$ :  $I, 5 = 2.8 k^{2} \times 69.8$ , k = 0.0876 m or k = 87.6 mm(b)  $ZF_{t} = m\bar{r}\alpha$ : 2F = 2.8(0.0240)69.8  $\frac{F = 2.35 \text{ N}}{F = 2.35 \text{ N}} = 0.0240 \text{ m}$ (c)  $ZF_{n} = m\bar{r}\omega^{2}$ :  $2R = 2.8(0.0240)(125.7^{2})$   $\frac{R}{60} = 125.7 \text{ rad/s} = 0.0240 \text{ m}$ 



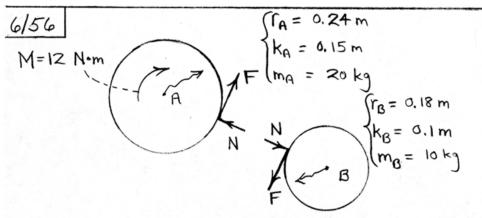




6/53 Rim:  $I_0 = mr^2 = \frac{100}{32.2} \left(\frac{18}{12}\right)^2 = 6.99 \text{ lb-ft-sec}^2$ Each spoke:  $I_0 = \frac{1}{3}mL^2 = \frac{1}{3}\frac{15}{32.2}\left(\frac{18}{12}\right)^2 = 0.349 \text{ lb-ft-sec}^2$   $= 0.349 \text{ lb-ft-sec}^2$   $\geq M_0 = I_0 \propto \frac{400}{12} = \left[6.99 + 3(0.349)\right] \propto \propto = 4.15 \text{ rad/sec}^2$   $\geq F_{\pm} = \sum_{i} m \vec{r} \vec{d}; \ 0 = \frac{15}{32.2} \left(\frac{9}{12}\right)(4.15) \neq 0 \quad (middle spoke only)$   $O_{\pm} = 1.449 \text{ lb}$ 



| 6/55 | For entire assembly, |  $I_{ZZ} = 0.60 + (0.080 + 12(0.2)^2) = 1.160 \text{ kg} \cdot \text{m}^2$  |  $ZM_Z = I_{ZZ} \propto : 16 = 1.160 \propto, \propto = 13.79 \text{ rad/s}^2$  | For Cylinder: |  $ZF_t = ma_t : A + B = 12(0.2)(13.79)$  |  $ZM_0 = I_{ZZ} \propto : M_0 = I_{ZZ} \sim : M_0 = I_{ZZ} \propto : M_0 = I_{ZZ} \sim : M_0 = I_{ZZ$ 



$$+^{\gamma} \sum M_{A} = I_{A} \propto_{A} : 12 - F(0.24) = 20(0.15)^{2} \propto_{A} (1)$$

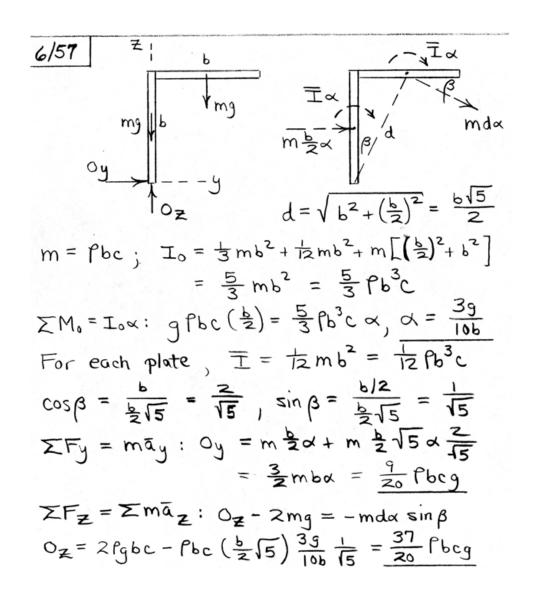
$$F = I_{B} \propto_{B} : F(0.18) = 10(0.1)^{2} \propto_{B} (2)$$

Tangential accelerations match: 
$$r_A \propto_A = r_B \propto_B$$
  
0.24  $\propto_A = 0.18 \propto_B$  (3)

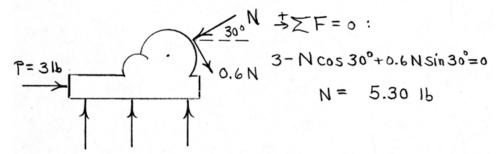
Solution of Eqs. (1)-(3): 
$$F = 14.16 \text{ N}$$

$$\alpha_{A} = 19.12 \text{ rod/s}^{2}(CW)$$

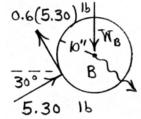
$$\alpha_{B} = 25.5 \text{ rod/s}^{2}(CCW)$$



6/58 Power unit C:



Wheel B:  $12 \sum M_B = I_B x$ : 0.6(5.30)( $\frac{10}{12}$ ) =  $\frac{50}{32.2} (\frac{8}{12})^2 x$ 



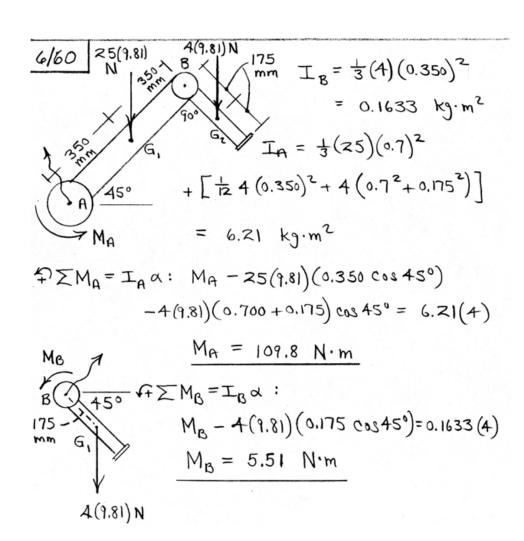
 $\alpha = 3.84 \text{ rad/sec}^2$ 

Steady-state speed:

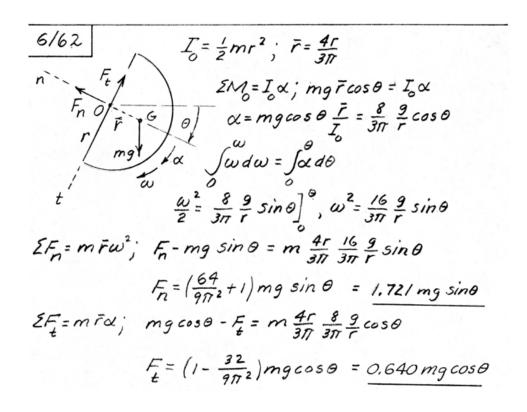
$$\omega_{B} = \frac{r_{B} \omega_{B}}{r_{B}} = \frac{8 \left[1600 \frac{2\pi}{60}\right]}{10}$$

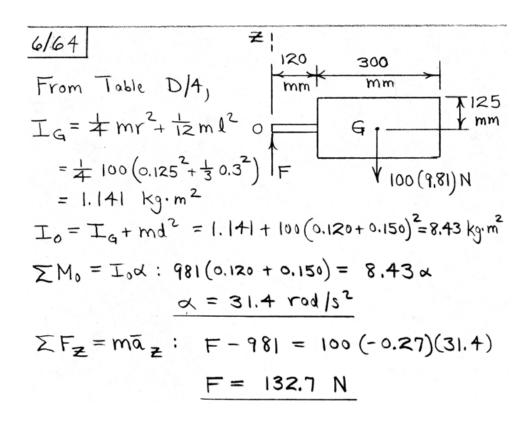
$$\omega_{B} = \omega_{Bo}^{10} + \alpha t$$
:  $t = \frac{\omega_{B}}{\alpha} = \frac{134.0 \text{ rad/sec}}{3.84} = 34.9 \text{ sec}$ 

6/59 For complete ring  $I_0 = 2(2m)r^2$ For the half ring  $I_0 = 2mr^2$   $I_0 = I_0 \alpha$ ;  $M = 2mr^2 \alpha$ ,  $\alpha = \frac{M}{2mr^2}$   $I_0 = I_0 \alpha$ ;  $I_0 = I_0 \alpha$ ;  $I_0 = I_0 \alpha$   $I_0 = I_0 \alpha$ ;  $I_0 = I_0 \alpha$ ;  $I_0 = I_0 \alpha$   $I_0 = I_0 \alpha$ ;  $I_0 = I_0 \alpha$ ;  $I_0 = I_0 \alpha$   $I_0 = I_0 \alpha$ ;  $I_0 = I_0 \alpha$ ; I

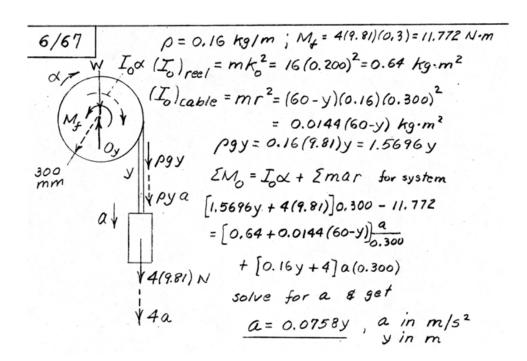


 $\frac{G/61}{t}$   $M \quad (case b)$   $\frac{1}{t}$   $M \quad (case b)$   $M \quad (case b)$   $M \quad (case b)$   $M \quad (case a)$   $M \quad (case b)$   $M \quad (case a)$   $M \quad (case a$ 





6/66  $M = \frac{T}{2}\Gamma$ ,  $T = \frac{2(900)}{2} = 900 16$ t.  $2M_0 = I_0 \alpha$ ;  $T = \frac{2000}{2} = 900 16$   $2M_0 = I_0 \alpha$ ;  $1 = \frac{1}{3} = \frac{600}{32.2} = \frac{72}{2} \alpha$   $1 = \frac{1}{3} = \frac{600}{32.2} = \frac{72}{2} \alpha$   $2 = 0.0899 \text{ rad/sec}^2$   $2 = \frac{1}{3} = \frac{600}{32.2} = \frac{600}{3$ 

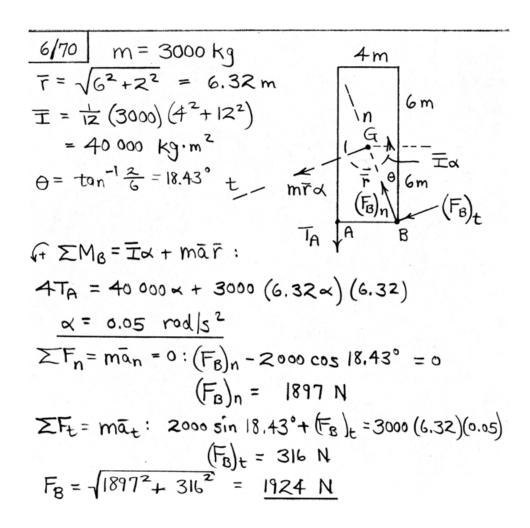


6/68 
$$I = \frac{1}{3} m \ell^2 + (\frac{1}{12} m \ell^2 + m \ell^2) = \frac{17}{12} (8)(0.5)^2$$
 $R_1 = \frac{1}{3} m \ell^2 + (\frac{1}{12} m \ell^2 + m \ell^2) = \frac{17}{12} (8)(0.5)^2$ 
 $R_2 = \frac{1}{3} m \ell^2 + (\frac{1}{12} m \ell^2 + m \ell^2) = \frac{17}{12} (8)(0.5)^2$ 
 $R_3 = \frac{1}{3} m \ell^2 + (\frac{1}{12} m \ell^2 + m \ell^2) = \frac{17}{12} (8)(0.5)^2$ 
 $R_4 = \frac{1}{3} m \ell^2 + (\frac{1}{12} m \ell^2 + m \ell^2) = \frac{17}{12} (8)(0.5)^2$ 
 $R_4 = \frac{1}{3} m \ell^2 + (\frac{1}{12} m \ell^2 + m \ell^2) = \frac{17}{12} (8)(0.5)^2$ 
 $R_4 = \frac{1}{3} m \ell^2 + (\frac{1}{12} m \ell^2 + m \ell^2) = \frac{17}{12} (8)(0.5)^2$ 
 $R_5 = \frac{17}{12} (8)(0.5)^2$ 
 $R_6 = \frac{17}{12} (8)(0.5)^2$ 

6/69 Beam, 
$$I_0 = \frac{1}{3} \frac{2000}{32.2} I_0^2 + \frac{500}{32.2} (2^2 + 4^2)$$

The second second

The mrd resultant for the winch has an x-component so that Q = 0.



$$\frac{6/71}{x} = \frac{\sin (45^{\circ}-\theta)}{x} = \frac{\sin 135^{\circ}}{1}$$
Differentiate WRT time t:
$$-\theta \cos (45^{\circ}-\theta) + \theta^{2} \sin (45^{\circ}-\theta) = \frac{\sin 135^{\circ}}{1}$$

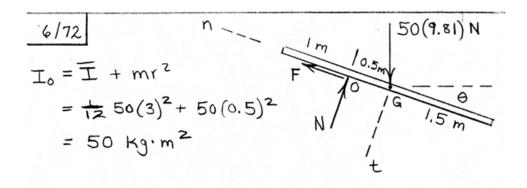
$$-\theta \cos (45^{\circ}-\theta) + \theta^{2} \sin (45^{\circ}-\theta) = \frac{\sin 135^{\circ}}{1}$$
So  $\theta = -\frac{x}{1} \frac{\sin 135^{\circ}}{\cos (45^{\circ}-\theta)}$ ; For  $1 = 8 \text{ m} \stackrel{?}{=} \theta = 35^{\circ}$ ,
and  $x = 3 \text{ m/s}^{2}$   $\theta = -0.275 \text{ rod/s}^{2}$  (CW)
$$mg = 120(9.81) = 1177 \text{ KN}$$

$$T_{A} = \frac{1}{3} \text{ m} (b^{2} + 1^{2})$$

$$= \frac{1}{3} 120(10^{3}) \left[3^{2} + 8^{2}\right]$$

$$= 2920 (10^{3}) \text{ kg·m}^{2}$$

$$+ \sum_{B} \sum_{A} \sum_{B} \sum_{B}$$



$$\sum M_{o} = I_{o} \propto : 50(9.81)(0.5 \cos \theta) = 50 \propto$$

$$\propto = 4.905 \cos \theta = \omega \frac{d\omega}{d\theta}, \quad \int_{0}^{\omega} \omega d\omega = \int_{0}^{\theta} 4.905 \cos \theta d\theta$$

$$\omega^{2} = 9.81 \sin \theta$$

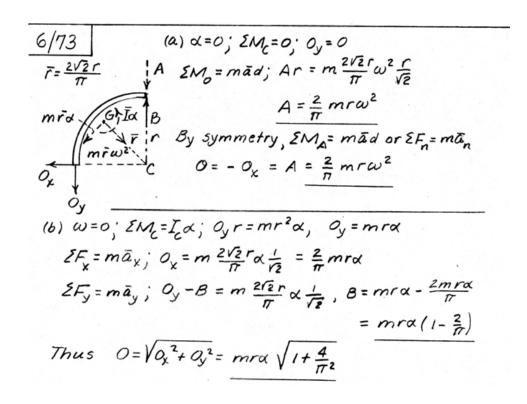
$$\sum F_{t} = m\bar{\alpha}_{t} : 50(9.81) \cos \theta - N = 50(0.5)(4.905 \cos \theta)$$

$$\sum F_{n} = ma_{n} : F - 50(9.81) \sin \theta = 50(0.5)(9.81 \sin \theta)$$

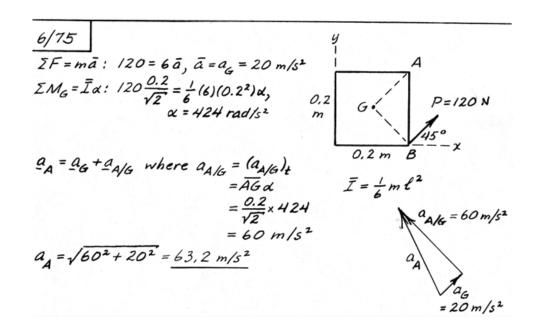
$$Slipping occurs when  $F = 0.30N$ 

$$Z^{nd} = 9: 0.3N = 75(9.81) \sin \theta$$

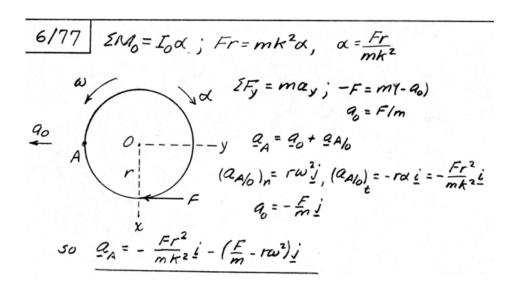
$$Divide + 0.30 =$$$$



6/74  $2M_0 = I_0 \lambda; \ mg \frac{l}{2} \sin \theta = \frac{1}{3}ml^2 \alpha$   $\alpha = \frac{3g}{2l} \sin \theta$   $\omega \qquad \theta \qquad \omega^2 = \frac{3g}{l} (-\cos \theta)^{\theta}$   $\omega^2 = \frac{3g}{l} (1-\cos \theta)$   $mg \qquad \Sigma F_n = m\tilde{\alpha}_n; \ mg \cos \theta - N = m \frac{l}{2} \omega^2$   $N = mg \cos \theta - m \frac{l}{2} \frac{3g}{l} (1-\cos \theta)$   $= mg \left[\cos \theta - \frac{3}{2} (1-\cos \theta)\right] = \frac{mg}{2} (5\cos \theta - 3)$   $\Sigma F_t = m\tilde{\alpha}_t; \ mg \sin \theta - F = m \frac{l}{2} \frac{3g}{2l} \sin \theta, \ F = \frac{mg}{4} \sin \theta$   $(a) \text{Slips at } \theta = 30^\circ, \ \mu_s = F/N = \frac{mg \sin 30^\circ/4}{mg (5\cos 30^\circ - 3)} = \frac{0.188}{mg (5\cos 30^\circ - 3)}$   $(b) \text{No slip: } N = 0 \text{ when } \cos \theta = 3/5, \ \theta = 53.1^\circ$ 



$$\begin{split} \mathcal{E}F_{\chi} = m\bar{a}_{\chi}; \; 3 = \frac{64.4}{32.2}a, \; a = 1.5 \; \text{ft/sec}^2 \\ \mathcal{U}^{=2} = 2\alpha\chi, \; \mathcal{U}^{2} = 2(1.5)(3) = 9 \\ \mathcal{U} = 3 \; \text{ft/sec} \\ \mathcal{E}M_{G} = \bar{I}\alpha; \; 3\frac{10}{12} = \frac{1}{2} \frac{64.4}{32.2} \frac{10}{12} \alpha \\ \mathcal{U} = 3.6 \; \text{rad/sec} \\ \mathcal{U} = \alpha t; \; \mathcal{U} = 3.6 \; (2) = 7.2 \; \text{rad/sec} \end{split}$$



6/78  $I = m\bar{k}^2 = 300 (1.5)^2 = 675 \text{ kg} \cdot m^2$   $I = m\bar{k}^2 = 300 (1.5)^2 = 300 (1.5)^2 = 300 (1.5)^2$   $I = m\bar{k}^2 = 300 (1.5)^2 = 300 (1.5)^2 = 300 (1.5)^2$   $I = m\bar{k}^2 = 300 (1.5)^2 = 300 (1.5)^2 = 300 (1.5)^2$   $I = m\bar{k}^2 = 300 (1.5)^2 = 300 (1.5)^2 = 300 (1.5)^2 = 300 (1.5)^2 = 300 (1.5)^2 = 300 (1.5)^2 = 300 (1.5)^2 = 300 (1.5)^2 = 300 (1.5)^2 = 300 (1.5)^2 = 300 (1.5)^2 = 300 (1.5)^2 = 300 (1.5)^2 = 300 (1.5)^2 = 300 (1.5)^2 = 300 (1.5)^2$ 

$$\frac{6/79}{40^{\circ}} + \frac{y}{40^{\circ}}$$

$$\begin{cases} mg = 8 \text{ lb}, \quad \overline{I} = \frac{1}{2}mr^{2} \\ \mu_{S} = 0.3, \quad \mu_{K} = 0.20 \\ \theta = 40^{\circ} \end{cases}$$

$$N \text{ Vmg} = 8 \text{ lb}$$

$$\Sigma F_{\chi} = m \bar{a}_{\chi} : -F + 8 \sin 40^{\circ} = \frac{8}{32.2} a$$
 (1)

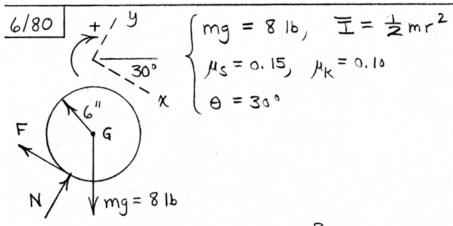
$$\Sigma Fy = 0$$
:  $N - 8\cos 40^\circ = 0$  (2)

$$\sum M_G = \overline{I} \propto : F\left(\frac{6}{12}\right) = \frac{1}{2} \frac{8}{32.2} \left(\frac{6}{12}\right)^2 \propto (3)$$

Assume rolling with no slip: 
$$a = \frac{6}{12} \alpha$$
 (4)

Solution of (1) - (4): 
$$F = 1.714 \text{ 1b}$$
  $\alpha = 13.80 \frac{\text{ft}}{\text{Sec}^2}$   
 $N = 6.13 \text{ 1b}$   $\alpha = 27.6 \frac{\text{rad}}{\text{sec}^2}$ 

$$F_{\text{max}} = \mu_S N = 0.3 (6.13) = 1.839 \text{ lb > F}$$
  
Assumption valid.



$$\Sigma F_{\chi} = m \bar{a}_{\chi} : -F + 8 \sin 30^{\circ} = \frac{8}{32.2} \alpha$$
 (1)

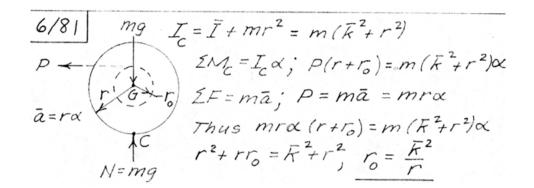
$$\sum F_y = 0 : N - 8 \cos 30^\circ = 0$$
 (2)

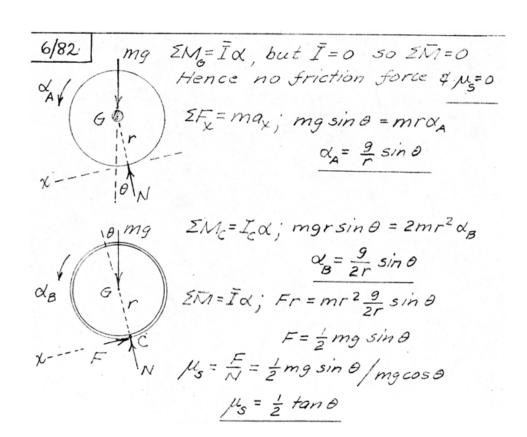
$$\sum M_{G} = \overline{L} \alpha : F\left(\frac{6}{12}\right) = \frac{1}{2} \frac{8}{32.2} \left(\frac{6}{12}\right)^{2} \alpha \qquad (3)$$

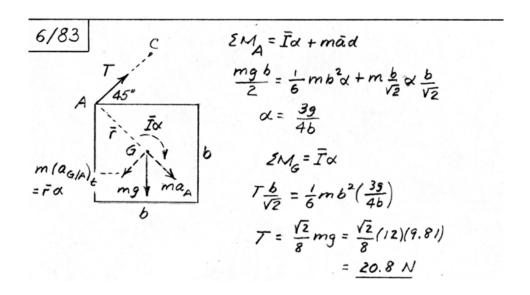
Assume rolling with no slip: 
$$a = \frac{6}{12} \propto$$
 (4)

Solution of (1) - (4): 
$$F = 1.333$$
 lb  $\alpha = 10.73 \frac{ft}{sec^2}$   
 $N = 6.93$  lb  $\alpha = 21.5 \frac{rod}{sec^2}$ 

F= 
$$\mu_k N = 0.10(6.93) = 0.693 \, lb$$
  
From Eqs. (1)  $4(3)$ :  $\alpha = 13.31 \, ft/sec^2$ ,  $\alpha = 11.15 \, rad/sec^2$ 







$$6/84$$

$$Q_{A} = 0$$

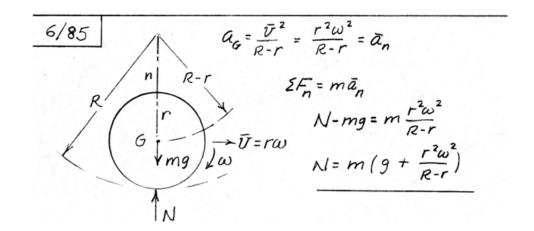
$$W_{AB} = 0$$

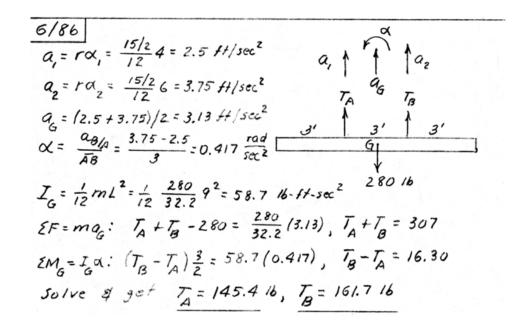
$$\overline{a} = \frac{U^{2}}{2r}$$

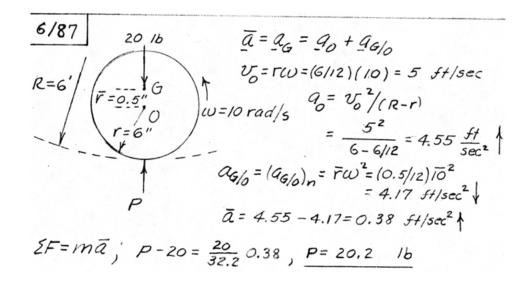
$$A = m\overline{a} = \frac{U^{2}}{2r}$$

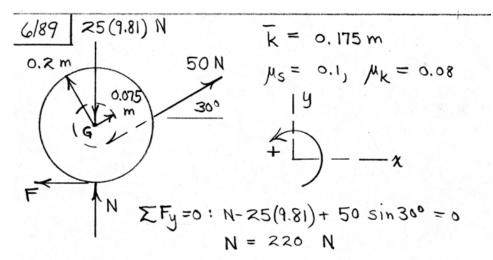
$$A = m \frac{U^{2}}{2r} + \overline{a} = 0$$

$$A = m \frac{U^{$$









$$\Sigma F_{\chi} = m \bar{a}_{\chi} : 50 \cos 30^{\circ} - F = 25a$$
 (1)

$$\sum M_G = I \propto : 50(0.075) - F(0.2) = 25(0.175)^2 \propto (2)$$

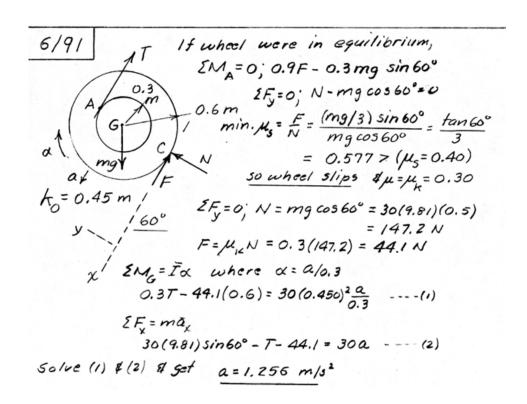
Assume rolling with no slip: 
$$\alpha = -rd$$
 (3)  
Solution of (1)-(3):  $\alpha = 0.556 \text{ m/s}^2$ ,  $\alpha = -2.78 \text{ rad/s}^2$   
 $\alpha = -2.78 \text{ rad/s}^2$ 

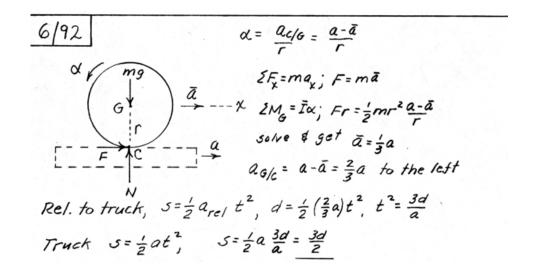
 $F_{\text{max}} = \mu_s N = 0.1 (220) = 22.0 \text{ N} < F: Slips, \frac{F = \mu_k N = 17.62 N}{200}$ From Eqs. (1) \(\frac{1}{2}\): \(\alpha = 1.027 \text{ m/s}^2\) \(\alpha = 0.295 \text{ rad/s}^2\)

$$\sum F_{\chi} = m\bar{\alpha}_{\chi}$$
: 30 cos 70° - F = 25a (1)

$$\sum M_G = \bar{I}_{\alpha} : 30(0.075) - F(0.2) = 25(0.175)^2 \alpha$$
 (2)

Solution of Eqs. (1)-(3): 
$$\begin{cases} \alpha = -0.0224 \text{ m/s}^2 \\ \alpha = 0.1121 \text{ rad/s}^2 \\ F = 10.82 \text{ N} \end{cases}$$





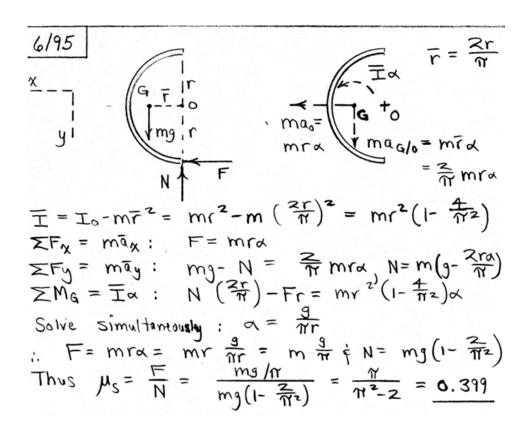
6/93

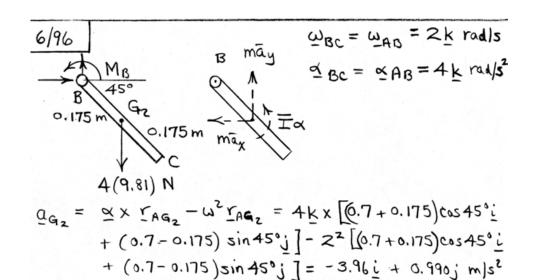
Ay

Ax

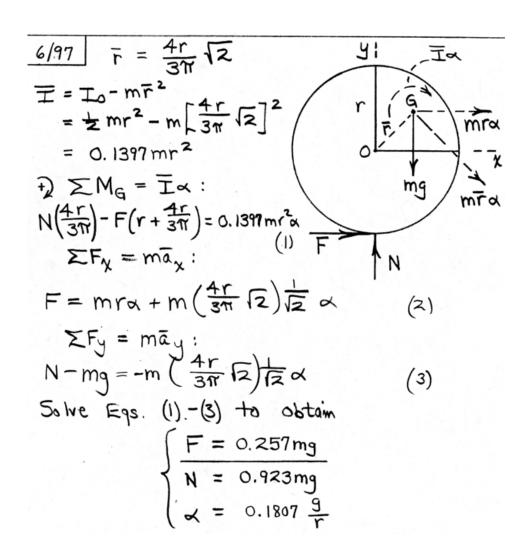
0.4m

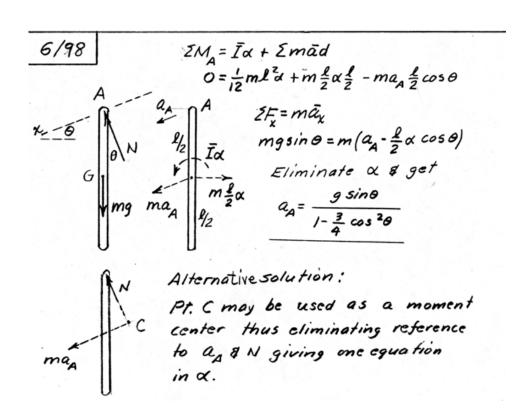
$$G = \{ -\frac{G}{4} \} \}$$
 $m = \{ -\frac{G}{4} \} \}$ 
 $m = \{ -\frac{G}{4} \}$ 
 $m = \{ -\frac{G}{4} \} \}$ 
 $m = \{ -\frac{G}{4} \}$ 
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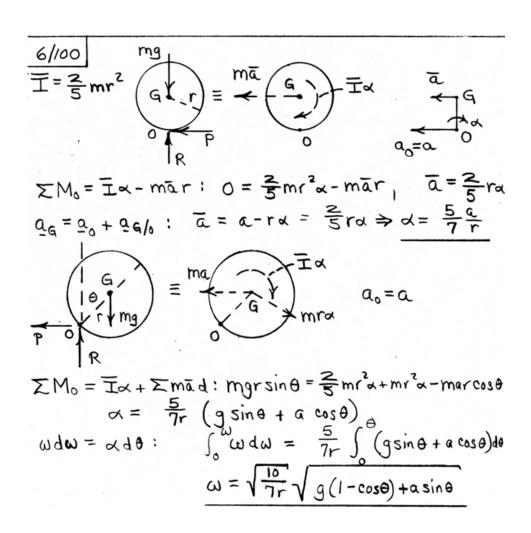


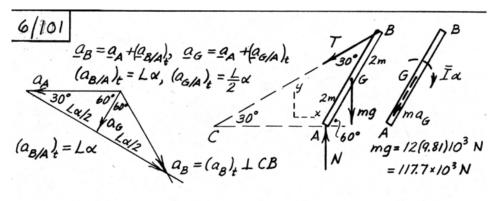
$$\sum M_B = \overline{I} \times + \sum mad : M_B - 4(9.81)(0.175 \sin 45^\circ) = \frac{1}{12}(4)(0.35)^2(4) + 4(0.990)(0.175 \cos 45^\circ) - 4(3.96)(0.175 \sin 45^\circ), M_B = 3.55 N·m (CCV)$$





 $\frac{6/99}{C_{1}} = \frac{T_{2}}{C_{1}} = \frac{T_{2}}{C_{1}} = \frac{g}{w_{2}} = \frac{g}{k^{2}/r^{2}+1} = \frac{g}{k^{2}/r^{2}+1}$ 



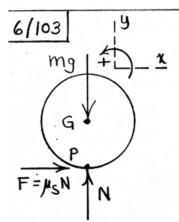


From accel. diag.  $a_B = \frac{L}{2} \alpha \sec 30^\circ = \frac{4}{2} \sec 30^\circ d = 2.30 \alpha \text{ m/s}^2$ Since  $a_G$  passes through A,

 $\Sigma M_A = \bar{I}_{\alpha}: 1/7.7(10^3)2\cos 60^\circ - T \times 4\sin 30^\circ = \frac{1}{12}12(10^3)4^2 \alpha$  $1/7.7(10^3) - 2T = 16(10^3)\alpha - (a)$ 

 $\Sigma F_{\chi} = m\bar{a}_{\chi}$ :  $T\cos 30^{\circ} = 12(10^{3})(a_{6})_{\chi}$  where  $a_{6} = \frac{L}{2}\alpha \tan 30^{\circ}$ ,  $(a_{6})_{\chi} = \frac{L}{2}\tan 30^{\circ}\alpha \cos 60^{\circ} = 0.577\alpha \frac{m}{s^{2}}$ 

so  $T = 8(10^3)\alpha$  --- (b) Solve (a) \$\frac{4}{5}\$ (b) \$\frac{4}{9}\$ et \$\alpha = 3.68\$ rad/s², T = 29.4 kN  $a_A = \frac{L}{2}\alpha/\cos 30^\circ = \frac{4}{2}(3.68)/\cos 30^\circ$ ,  $a_A = 8.50$  m/s² Clears the Surface is very small and that the speed of B is constant (while on surface). Time t between  $A \neq B$  leaving surface:  $t = \frac{1}{V}$ .  $T = 2m(\frac{1}{2})^2 = ml^2/2$   $T_B = \frac{ml^2}{2} + 2m(\frac{1}{2})^2 = ml^2$   $Z_{Mg} = T_B \ddot{\theta} : Z_{Mg} \dot{Z} = ml^2 \ddot{\theta} , \ddot{\theta} = \frac{9}{1} (CCW)$   $\omega = \omega_0 + \ddot{\theta} \dot{t} = \frac{9}{1} \dot{V} = \frac{9}{V}$ 



The no-slip constraint is found by equating the horizontal acceleration of point P to the acceleration ac of the cart

$$(a_P)_{hor} = a_G + r\alpha = a_C$$
 (1)

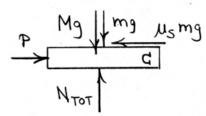
$$\sum F_y = 0 \Rightarrow N = mg$$
 (2)

$$\Sigma F_{\chi} = ma_{G}: \mu_{S} m_{g} = ma_{G}$$
 (3)

$$\Sigma F_{\chi} = ma_{G}: \mu_{S} m_{g} = ma_{G}$$
 (3)  
 $\Sigma M_{G} = \Xi \alpha : + \mu_{S} m_{g} r = m k^{2} \alpha$  (4)

Solution of (1), (3),  $\dot{\epsilon}(4)$ :  $\begin{cases} a_G = \mu_S g \\ a_C = \mu_S g \left[ 1 + \frac{r^2}{K^2} \right] \\ \alpha = + \mu_S g r / \bar{k}^2 \end{cases}$ 

Cart:



$$\sum F_{\chi} = M a_{c}:$$

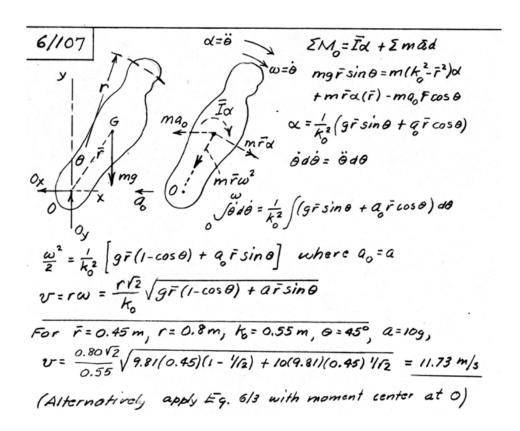
$$P - \mu_{s} m_{g} = M \mu_{s} g \left[ 1 + \frac{r^{2}}{k^{2}} \right]$$

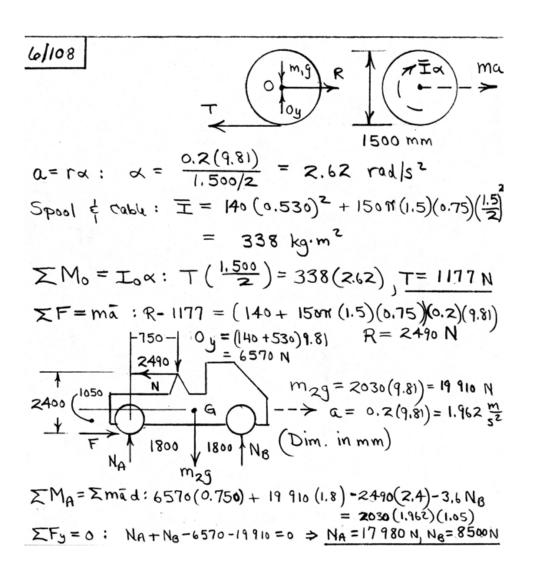
$$P = \mu_{s} g \left[ m + M \left( 1 + \frac{r^{2}}{k^{2}} \right) \right]$$

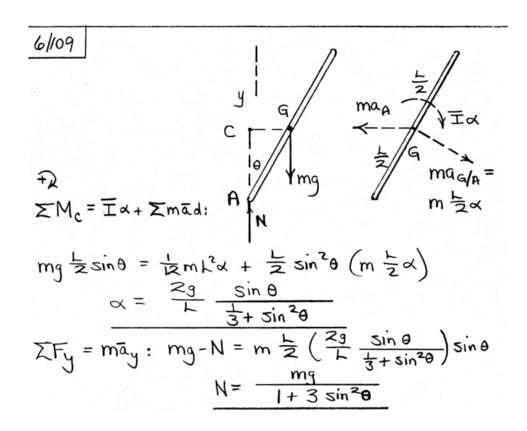
6/104  $m\bar{a} = mq_0 + mq_{6/6} = mq + m\bar{r}\omega^2 + m\bar{r}\alpha$   $a = 3 \text{ ft/s}\alpha^2, \bar{r} = 6 \text{ ft}$   $a = 3 \text{ ft/s}\alpha^2, \bar{r} = 6 \text{ ft}$   $a = 3 \text{ ft/s}\alpha^2, \bar{r} = 6 \text{ ft}$   $a = 3 \text{ ft/s}\alpha^2, \bar{r} = 6 \text{ ft}$   $a = 3 \text{ ft/s}\alpha^2, \bar{r} = 6 \text{ ft}$   $a = 3 \text{ ft/s}\alpha^2, \bar{r} = 6 \text{ ft}$   $a = 3 \text{ ft/s}\alpha^2, \bar{r} = 6 \text{ ft}$   $a = 4 \text{ ft/s}\alpha^2, \bar{r} = 6 \text{ ft/s}\alpha^2, \bar{r} =$ 

6/105  $I_{A} = \frac{1}{12}m([1.8]^{2} + [3.0]^{2}) + m([0.9]^{2} + [1.5]^{2})$   $y_{A} = \frac{4}{3}m([0.9]^{2} + [1.5]^{2}) = 4.08 \text{ m}$   $y_{A} = \frac{4}{3}m([0.9]^{2} + [1.5]^{2}) = 4.08 \text{ m}$   $y_{A} = \frac{1}{4}x + [5xma]$   $y_{A} = I_{A}x +$ 

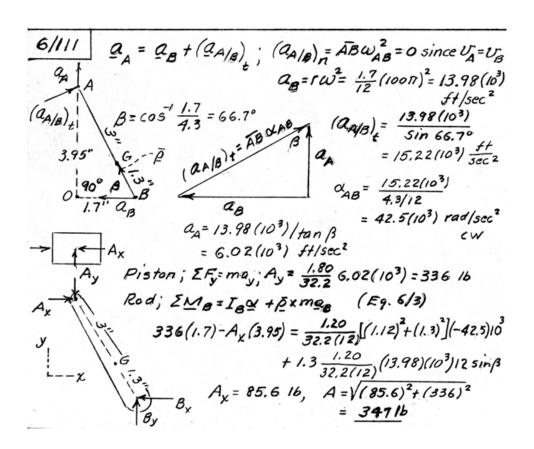
6/106  $\Sigma F = ma$ ;  $\mu W = \frac{W}{9}a$ ,  $a = \mu g$   $v = v_0 - \mu g t$ ,  $t = \frac{v_0 - v}{\mu g} = time$  to slow down to vel. v  $T = u = v_0 + u = v_0 +$ 

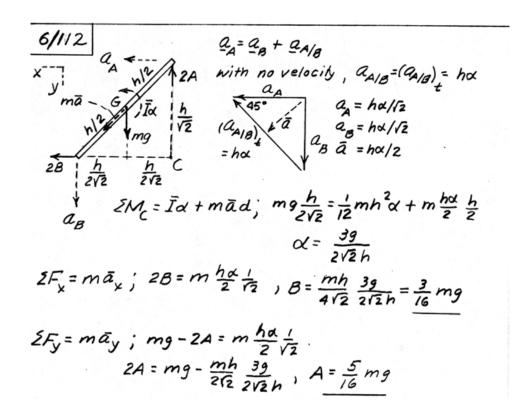




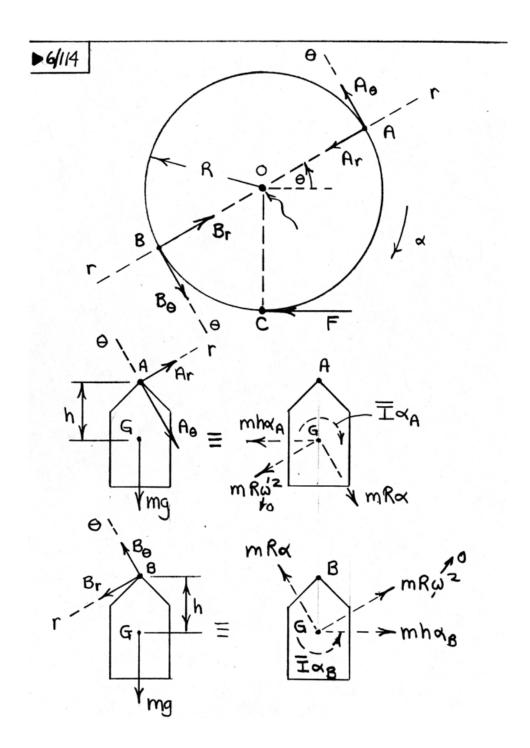


6/110  $\vec{r} = \vec{0}\vec{G} = 0.040 \text{ m}$ ;  $m\vec{r}\vec{w}^2 = 10 (0.040)(2^2) = 1.6 \text{ N}$   $\vec{W} = 2 \text{ rad/s}$   $\vec{I} = m\vec{k}^2 = 10 (0.064)^2 = 0.0410$   $m\vec{r}\vec{w}^2 = 0.1 \text{ m}$   $m\vec{r}\vec{w} = 10 (0.040) \frac{a_0}{o.1} = 4a_0 \text{ N}$   $\vec{M} = \vec{I}\vec{w} + \vec{E}\vec{m}\vec{a}\vec{d}$   $\vec{A} = 10 (0.040) = 0.0410 \frac{a_0}{o.1} + 4a_0 (0.040)$   $\vec{A} = 2.60 \text{ m/s}^2$   $\vec{A} = 2.60 \text{ m/s}^2$ 





 $\sum F_{x} = m\bar{a}_{x} : R_{A} + 6\cos 15^{\circ} + R_{B}\sin 15^{\circ} = \frac{8}{32.2} \bar{a}_{x} \qquad (1)$   $R_{A} = \frac{8}{32.2} \bar{a}_{x} \qquad (1)$   $\sum F_{y} = m\bar{a}_{y} : R_{B}\cos 15^{\circ} - 6\sin 15^{\circ} - 8 = \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (2)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (3)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (3)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (3)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (3)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (3)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (3)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (3)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (3)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{32.2} \bar{a}_{y} \qquad (3)$   $\sum M_{G} = \bar{I}_{x} : \frac{8}{3$ 



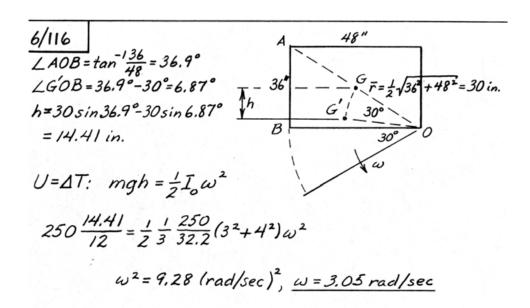
Substitute (1) \( \frac{1}{k} \) into (3)

FR - \( \frac{1}{2} \) \[ 2m Rx - 2m Rx \left( \frac{h \sin \theta\_n}{k} \right)^2 \] R = Iox

Simplify \( \frac{1}{k} \) Solve for F:  $F = \left\{ mR \left[ n - 2 \frac{h^2}{k^2} \left( \sin^2 \theta_1 + \sin^2 \theta_2 + \cdots \sin^2 \theta_{n/2} \right) \right] + \frac{Io}{R} \right\} x$ (n = 0 corresponds to \theta = 0; \( n/2 \) corresponds to \theta < \text{T} \)

Note: The above expression for F simplifies to  $F = \left\{ mRn \left( 1 - \frac{h^2}{2k^2} \right) + \frac{Io}{R} \right\} x$ 

6/115 
$$I_0 = \frac{1}{12}ml^2 + m(\frac{1}{4})^2 + 2m(\frac{3l}{4})^2$$
  
 $= \frac{6l}{48}ml^2$   
 $T_1 + U_{1-2} = T_2: 0 + mg(\frac{1}{4}) + 2mg(\frac{3l}{4}) = \frac{1}{2}\frac{6l}{48}ml^2\omega^2$   
 $\omega = 1.660\sqrt{\frac{3}{1}}$  CW



Weight cancels so does not influence the results.

 $\frac{6/117}{T_1 + U_{1-2}} = T_2$   $T_1 = \frac{1}{2} 8(0.3)^2 + \frac{1}{2} 12(0.210)^2 \left(\frac{0.3}{0.2}\right)^2 = 0.955 J$   $U_{1-2} = 8(9.81)(1.5) - 3\left(\frac{1.5}{0.2}\right) = 95.2 J$   $T_2 = \frac{1}{2} 8 y^2 + \frac{1}{2} 12(0.210)^2 \left(\frac{y}{0.2}\right)^2 = 10.62 y^2$ So  $0.955 + 95.2 = 10.62 y^2$ , y = 3.01 m/s

6/118  $U_{1-2} = \Delta T + \Delta V_g$   $0 = \frac{1}{2}m(4^2 - 0^2) - m_g(5)(1 - \cos \theta)$  $\theta = 33.2^{\circ}$ 

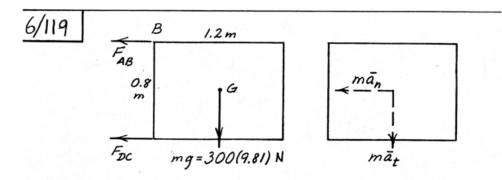
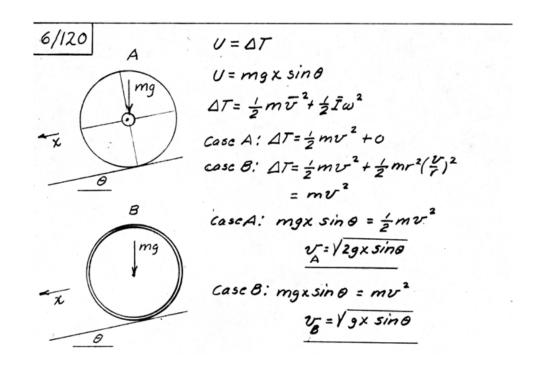


Plate has curvilinear translation so  $T = \frac{1}{2}mv^2$ 

 $U = \Delta T$ : 300(9.81)(0.8 cos 60°) =  $\frac{1}{2}$ (300) $\sigma^2$ ,  $\sigma = 2.80$  m/s  $\omega = \sigma/r$ : Angular relocity of links is  $\omega = 2.80/0.8 = 3.50$  rad/s

$$\Sigma F_t = m\bar{a}_t$$
:  $\bar{a}_t = 9.81 \text{ m/s}^2$   
 $\bar{a}_n = \sigma^2/r$ :  $\bar{a}_n = 2.80^2/0.8 = 9.81 \text{ m/s}^2$ 

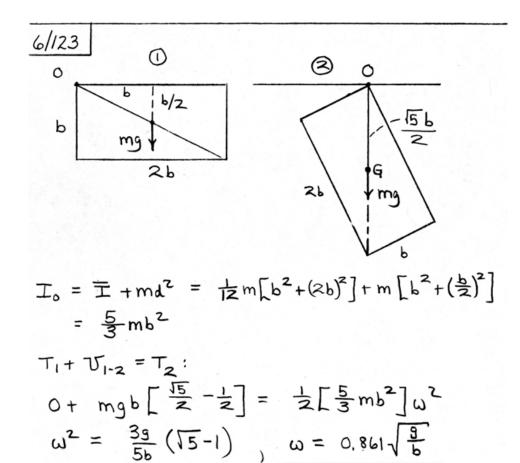
$$ZM_B = m\bar{a}d: 300(9.81)(0.6) + F_{DC}(0.8)$$
  
= 300(9.81)(0.6) + 300(9.81)(0.4)  
 $F_{DC} = 1472 \text{ N}$ 



6/121  $U = \Delta T$ ;  $U = 40(2 \times 3) = 240 \text{ J}$   $\Delta T_{hoop} = \frac{1}{2} m \bar{u}^2 + \frac{1}{2} \bar{I} \omega^2 = \frac{1}{2} m \omega^2 (r^2 + r^2) = m r^2 \omega^2$   $= 10(0.3)^2 \omega^2 = 0.9 \omega^2$   $\Delta T_{each pair} = \frac{1}{2} m \bar{v}^2 + \frac{1}{2} \bar{I} \omega^2 = \frac{1}{2} m r^2 \omega^2 + \frac{1}{2} \frac{1}{12} m (2r)^2 \omega^2$   $= \frac{2}{3} m r^2 \omega^2$   $\Delta T_{both pair} = \frac{4}{3} m r^2 \omega^2 = \frac{4}{3} 4(0.3)^2 \omega^2 = 0.48 \omega^2$ Thus  $240 = 0.9 \omega^2 + 0.48 \omega^2$ ,  $\omega^2 = 173.9$   $\omega = 13.19 \, rad/s$ 

6/122

For rotation:  $T_{rot} = \frac{1}{2}I_c\omega^2 = \frac{1}{2}(4)(\frac{1}{12}mb^2 + m\frac{b}{2})\omega^2$   $= \frac{2}{3}mb^2\omega^2$ For translation:  $T_{tran} = \frac{1}{2}(4m)\upsilon^2 = 2m\upsilon^2$ For  $T_{tran} = T_{rot}$ :  $2m\upsilon^2 = \frac{2}{3}mb^2\omega^2$   $\upsilon = \frac{b\omega}{\sqrt{3}}$ 



Final position of OADE (all bars)

$$b = \omega \qquad \Delta V_g = 8bpg(-2b) = -16pgb^2$$

$$A \rightarrow V_A \qquad \Delta T = 8\left(\frac{1}{3}mb^2\right)\omega^2$$

$$2b \quad \forall \omega \qquad = \frac{8}{3}pb^3\omega^2$$

$$U = O = \Delta T + \Delta V_g$$

$$U = O = \frac{8}{3}pb^3\omega^2 - 16pgb^2$$

$$\omega^2 = 6g/b, \quad \omega = \sqrt{6g/b}$$

6/125 Note: the wheel has no motion in initial or final positions so  $\Delta T_{wheel} = 0$   $U' = \Delta V_g + \Delta T'$ ;  $U' = Fb \sin \theta$   $\Delta V_g = -2m_g g \frac{b}{2} \sin \theta$   $\Delta T = 2(\frac{1}{2} I_c \omega^2) = \frac{1}{3} m_o b^2 \omega^2$ Thus  $Fb \sin \theta = -m_o g b \sin \theta + \frac{1}{3} m_o b^2 \omega^2$   $\omega = \sqrt{\frac{3(F + m_o g) \sin \theta}{m_o b}}$ 

6/126 Power  $P = \frac{d(Energy)}{dt} = \frac{\Delta E}{t}$   $\Delta E = \frac{1}{2} \tilde{\Gamma}(\omega_2^2 - \omega_1^2) = \frac{1}{2} (1200)(0.4)^2 (5000)^2 - [3000]^2 ) (\frac{217}{60})^2$   $= 16.84(10^6) \text{ J}$   $P = \frac{16.84(10^6)}{2(60)} = 140.4(10^3) \text{ J/s or } W$ 50  $P = 140.4 \text{ KW} \text{ or } P = \frac{140.4(10^3)}{7.457(10^2)} = 188 \text{ hp}$ 

6/127  $\Delta V_g + \Delta T = 0$   $\Delta V_g = -5.4(3.08)(9.81)(3.3) = -538 \text{ J}$   $\Delta T = \frac{1}{2}6.0(3.08)(0.375 \text{ w})^2$   $+ \frac{1}{2} \left[ 41(0.30)^2 + (3.08)(18-6)(0.375)^2 \right] \omega^2$   $= 1.299 \omega^2 + 4.44 \omega^2 = 5.74 \omega^2$ Thus  $-538 + 5.74 \omega^2 = 0$ ,  $\omega^2 = 93.8$ ,  $\omega = 9.68 \text{ rad/s}$ 

6/128 
$$U'_{1-2} = \Delta T + \Delta V_g + \Delta V_e$$
  
 $U'_{1-2} = M\theta = \frac{\pi}{2}M = 1.571 M \text{ in.-1b}$   
 $\Delta T = \frac{1}{2}I_0\omega^2 - 0 = \frac{1}{2}(\frac{1/2}{32.2\times12}\times10^2)4^2 = 24.8 \text{ in.-1b}$   
 $\Delta V_g = Wh = 12(-8) = -96 \text{ in.-1b}$   
 $\Delta V_e = \frac{1}{2}k(\dot{x}_2^2 - \dot{x}_1^2) = \frac{1}{2}3([30 - 15\sqrt{2}]^2 - 0) = 115.8 \text{ in-1b}$   
Thus  $1.571M = 24.8 - 96 + 115.8$ ,  $M = 28.4 \text{ lb-in.}$ 

6/129 For system  $\Delta T + \Delta V_g = 0$  since U = 0Yoke:  $\Delta V_g = 3 \times 9.81(-0.3) = -8.83 \text{ J}$   $\Delta T = \frac{1}{2}I\omega^2 = \frac{1}{2}(3 \times 0.35^2)(\frac{V_A}{0.5})^2$   $= 0.735 V_A^2$   $A = \frac{1}{2}I_C\omega^2 = \frac{1}{2}(2 \times 4 \times 0.25^2)(\frac{V_A}{0.25})^2$   $= 4V_A^2$ Thus  $0.735 V_A^2 + 4V_A^2 - 8.83 - 19.62 = 0$   $4.735 V_A^2 = 28.45, V_A^2 = 6.01, V_A = 2.45 \text{ m/s}$ 

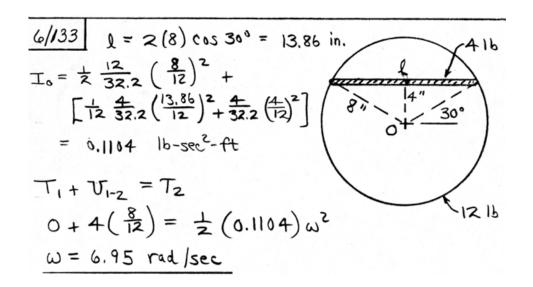
 $V_A = 0$ , so F and N are applied at a stationary point and thus do no work.

(b)  $v \neq 0$ ,  $v_A \neq 0$ : F and N do Work.

For the top position  $\omega_{8} = \frac{\sigma}{0.080}$ ,  $\omega_{0A} = \frac{\sigma}{0.280}$ For entire system  $U' = \Delta T + \Delta V_{g}$   $U'_{1-2} = M\theta = 4(\pi/2) = 6.28 \text{ J}$   $\Delta T_{0A} = \frac{1}{2}I_{0}\omega_{0A}^{2} = \frac{1}{2}0.8(0.140^{2})(\sigma/0.280)^{2} = 0.1\sigma^{2} \text{ J}$   $\Delta T_{B} = \frac{1}{2}m\sigma^{2} + \frac{1}{2}\bar{I}\omega^{2} = \frac{1}{2}0.9\sigma^{2} + \frac{1}{2}\left[\frac{1}{2}0.9\times0.080^{2}\right]\left(\frac{\sigma}{0.080}\right)^{2}$   $= 0.675\sigma^{2} \text{ J}$   $(\Delta V_{g})_{0A} = mgh = 0.8(9.81)(0.100) = 0.785 \text{ J}$   $(\Delta V_{g})_{8} = mgh = 0.9(9.81)(0.280) = 2.47 \text{ J}$ Thus  $6.28 = 0.1\sigma^{2} + 0.675\sigma^{2} + 0.785 + 2.47$ ,  $\sigma^{2} = 3.90 \text{ (m/s)}^{2}$   $\sigma = 1.976 \text{ m/s}$ 

$$\begin{array}{lll}
6/32 & T_{1} + U_{1-2} = T_{2} \\
T_{1} &= 0 \\
U_{1-2} &= \int_{1}^{\infty} M d\theta = \int_{0}^{\infty} Z(1 - e^{-0.1\theta}) d\theta \\
&= \left(2\theta + 20 e^{-0.1\theta}\right) \Big|_{0}^{5(2\pi)} \\
&= 2(5)(2\pi) + 20 e^{-0.1(5)(2\pi)} - 20 \\
&= 43.7 \text{ J}
\end{array}$$

$$T_{2} &= \frac{1}{2} T \omega^{2} = \frac{1}{2} (50)(0.4)^{2} \omega^{2} = 4\omega^{2}$$
So  $0 + 43.7 = 4\omega^{2}$   $\omega = 3.31 \text{ rad/s}$ 



6/134  $T = mk^2 = 10(0.090)^2 = 0.081 \text{ kg·m}^2$   $M = I\dot{\omega}$ ,  $\dot{\omega} = M/I = -2.10/0.081 = -25.9 \text{ rad/s}^2$   $\omega_0 = 80\,000(\frac{2\pi}{60}) = 8380 \text{ rad/s}$  $P = \frac{1}{4t}(\frac{1}{2}I\omega^2) = I\omega\dot{\omega}$ 

(a) 
$$t=0$$
:  $P = I\omega i = (0.081)(8380)(25.9)$   
= 17590 W or 17.59 KW

(b) 
$$t = 1205$$
:  $\omega = \omega_0 + \omega t = 8380 - 25.9$  (120)  
= 5270 rad/s  
 $P = T\omega \dot{\omega} = (0.081)(5270)(25.9) = 11060 \text{ W}$   
or  $P = 11.06 \text{ kW}$ 

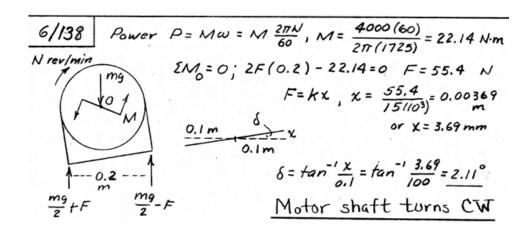
$$\omega_{\text{max}} = \omega_{\text{x}=0.211}$$

$$\omega_{\text{y}} = 0.789 \ell$$

$$\omega_{\text{only}} = 0.789 \ell$$

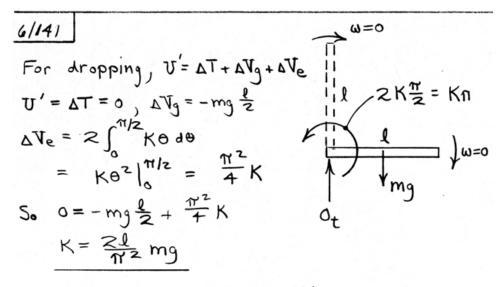
6//36  $O = \Delta V_g + \Delta T$ ;  $\Delta V_g = -200 \left[ \frac{12}{12} \sin 30^\circ + \frac{18}{12} (1 - \cos 30^\circ) \right]$   $\bar{K} = 4 \text{ in.}$  2 in. 2 in.

 $U = \Delta T : mg \left( \frac{8r}{3\pi} \right) = \frac{1}{2} I_c \omega^2$   $I_G = I_0 - m\bar{r}^2, I_c = I_G + m(r - \bar{r})^2$   $So \ I_c = I_0 - m\bar{r}^2 + m(r - \bar{r})^2$   $I_c = m \left( \frac{1}{2} r^2 - \bar{r}^2 + r^2 - 2r\bar{r} + \bar{r}^2 \right) = m \left( \frac{3}{2} r^2 - 2r \left[ \frac{4r}{3\pi} \right] \right) = mr^2 \left( \frac{3}{2} - \frac{8}{3\pi} \right)$   $So \ mg \left( \frac{8r}{3\pi} \right) = \frac{1}{2} mr^2 \left( \frac{3}{2} - \frac{8}{3\pi} \right) \omega^2, \ \omega^2 = \frac{32}{9\pi - 16} \frac{g}{r}, \ \omega = \sqrt{\frac{g}{r}} \frac{32}{9\pi - 16} \frac{rad}{s}$   $\bar{E}F_n = m\bar{a}_n : \ N - mg = m\bar{r} \omega^2, \ N = mg + m \frac{4r}{3\pi} \omega^2$   $N = mg \left( 1 + \frac{128}{3\pi (9\pi - 16)} \right)$ 

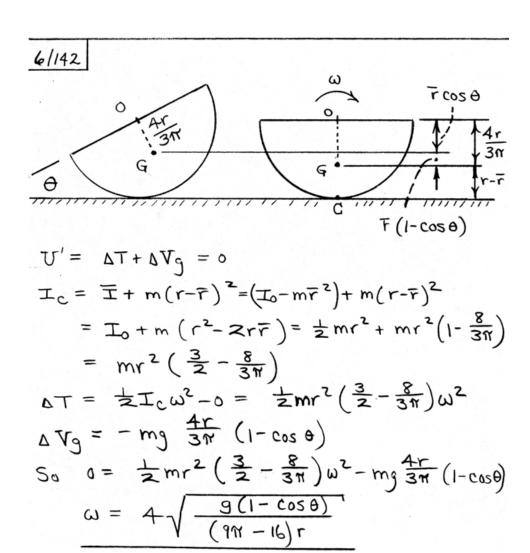


For treads  $T = 2(T_{hoop} + T_{top} section)$ ,  $T_{tottom} section = 0$   $T_{hoop} = \frac{1}{2}I_c\omega^2 = \frac{1}{2}\left[2\pi r\rho(r^2+r^2)\frac{v^2}{r^2}\right] = 2\pi \rho rv^2$   $T_{top} section = \frac{1}{2}(\rho b)(2v)^2 = 2\rho bv^2$   $U = M\theta = M\frac{s}{r}$   $Thus M\frac{s}{r} = 2(2\pi \rho rv^2 + 2\rho bv^2), M = 4\rho \frac{r}{s}v^2(\pi r + b)$ 

6/140  $\Delta V_{e} = \frac{1}{2}(1500) \left[ (0.1 + 2 \times 0.05)^{2} - \overline{0.1}^{2} \right] = 22.5 \text{ J}$   $\Delta V_{g} = -(150)(9.81)(0.05) = -73.58 \text{ J}$   $\Delta T = \sum_{e} \frac{1}{m} \overline{v}^{2} + \frac{1}{2} \overline{i} \overline{w}^{2} = \frac{1}{2}(150) \underline{v}^{2} + \frac{1}{2}(50)(0.3)^{2} \left( \frac{\underline{v}}{0.4} \right)^{2}$   $= 75 \underline{v}^{2} + 14.06 \underline{v}^{2} = 89.06 \underline{v}^{2}$   $\Delta T + \Delta V_{g} + \Delta V_{e} = 0; \quad 89.06 \underline{v}^{2} - 73.58 + 22.5 = 0$   $\underline{v}^{2} = 0.573, \quad \underline{v} = 0.757 \text{ m/s}$ 



Release from rest at  $\theta = \frac{\pi}{2}$ :  $\sum M_0 = I_0 \alpha$ :  $\frac{2!}{\pi^2} mg\pi - mg \frac{1}{2} = \frac{1}{3} ml^2 \alpha$ ,  $\alpha = 0.410 \frac{9}{2}$ Lid would not stay down;  $K = \frac{2!}{\pi^2} mg$  is not practical.



6/143 During rotation d0 of radial line,

disk rotates through angle dY between

lines OC' and O'C". CC'=CC" so  $Rd\theta = rd\beta$  &  $dY = d\theta + d\beta$   $= (1 + \frac{R}{r})d\theta$ or  $Y = (1 + \frac{R}{r})\theta$ For  $\theta = \frac{\pi}{3}$ ,  $Y = (1 + 0.6/0.15)\frac{\pi}{3} = 5\pi/3$  rad  $U' = \Delta T + \Delta V_g$ :  $U' = MY = 40\frac{5\pi}{3} = 209$  J  $\Delta T = \frac{1}{2}I_c\omega^2 = \frac{1}{2}(\frac{3}{2}mr^2)(\frac{\sigma}{r})^2 = \frac{3}{4}m\sigma^2 = \frac{3}{4}(30)\sigma^2$ 

 $U' = \Delta T + \Delta V_g: \quad U' = M\gamma = 40 \frac{5\pi}{3} = 209 J$   $\Delta T = \frac{1}{2} I_c \omega^2 = \frac{1}{2} \left( \frac{3}{2} m r^2 \right) \left( \frac{\sigma}{r} \right)^2 = \frac{3}{4} m \sigma^2 = \frac{3}{4} (30) \sigma^2$   $= 22.5 \sigma^2$   $\Delta V_g = mgh = mg(R+r) (1-\cos 60^\circ)$   $= 30(9.81)(0.75) \left( \frac{1}{2} \right) = 110.4 J$   $209 = 22.5 \sigma^2 + 110.4, \quad \sigma^2 = 4.40 \quad (m/s)^2, \quad \underline{\sigma} = 2.10 \quad m/s$ 

Total mass  $m = 2rf + 2\pi rf = 2rf(1+\pi)$ There f = mass per unit length.  $r = \frac{\sum rm}{\sum m} = \frac{2rf(r) + 2\pi rf(3r)}{2rf + 2\pi rf(3r)}$   $= r \frac{1+3\pi}{1+\pi}$   $= r \frac{1+3\pi}{1+\pi}$   $T_{B-B} = \frac{1}{3}(2rf)(2r)^2 + \left[2\pi rfr^2 + 2\pi rf(3r)^2\right]$   $= \frac{4+30\pi}{3(1+\pi)} mr^2$   $T_{A-B} = \frac{1}{3}(2rf)(2r)^2 + \left[\frac{1}{2}2\pi rfr^2 + 2\pi rf(3r)^2\right]$   $= \frac{8+57\pi}{6(1+\pi)} mr^2$   $T_1 + U_{1-2} = T_2$ (a)  $0 + mgr = \frac{1+3\pi}{1+\pi} = \frac{1}{2} \frac{8+57\pi}{6(1+\pi)} mr^2 \omega^2$   $\frac{\omega}{8+57\pi} = \frac{1}{2} \frac{3(1+\pi)}{3(1+\pi)} mr^2 \omega^2$   $\frac{\omega}{1+3\pi} = \frac{1}{2} \frac{3(1+\pi)}{3(1+\pi)} mr^2 \omega^2$ 

6/145 
$$P = \frac{dU}{dt} = \frac{d}{dt}(T + V_g) + RV$$

$$P = \frac{d}{dt} \left[ \frac{5}{2}mv^2 + \frac{1}{2}\bar{1}\omega^2 \right] + \frac{d}{dt}(mgh) + RV$$

$$= \sum mv \frac{dv}{dt} + \sum \bar{1}w \frac{dw}{dt} + mgv \sin\theta + RV$$

$$= mva + 4\bar{1}w + (mg\sin\theta + R)V$$

$$= (mv + 4\bar{1}\frac{v}{r^2})a + (mg\sin\theta + R)V$$
(a) with  $a = 0$ ,  $P = 0 + (500 \times 9.81 \times \frac{1}{\sqrt{101}} + 400)\frac{72}{3.6}$ 

$$= 17.761 \text{ W or } P = 17.76 \text{ kW}$$

(b) with 
$$a = 3 \text{ m/s}^2$$

$$P = \left(500 \frac{72}{3.6} + 4 (40)(0.4)^2 \frac{72/3.6}{\overline{0.6}^2}\right) 3 + 17.761$$

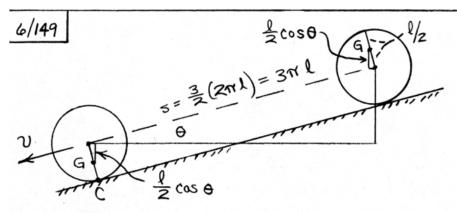
$$= 30.000 + 4.267 + 17.761$$

$$= 52.028 \text{ w or } P = 52.0 \text{ kW}$$

6/146  $\Delta T_{translational} = \frac{1}{2}mv^2 - 0 = \frac{1}{2}(10000)(\frac{72}{3.6})^2 - 0$   $= 2(10^6) \ J$   $\Delta T_{rotation} = \frac{1}{2}I(\omega_2^2 - \omega_1^2)$   $= \frac{1}{2}(1500)(0.5)^2(\omega_2^2 - \left[\frac{4000\times217}{60}\right]^2)$   $= 187.5\omega_2^2 - 32.90\times10^6 \ J$   $\Delta E = 0.1(187.5\omega_2^2 - 32.90\times10^6) = 18.75\omega_2^2 - 3.29\times10^6$   $\Delta V_g = mgh = 10000(9.81)(20) = 1.96\times10^6 \ J$   $\Delta E = \Delta T + \Delta V_g;$   $18.75\omega_2^2 - 3.29\times10^6 = 2\times10^6 + 187.5\omega_2^2 - 32.90\times10^6 + 11.96\times10^6 \ J$   $168.75\omega_2^2 = 25.65\times10^6; \quad \omega_2^2 = 152.000 \ (rad/s)^2$   $\omega_2 = 390 \ rad/s \ or \quad N = \frac{390\times60}{217} = \frac{3720 \ rev/min}{2}$ 

6/147 For equil.  $ZM_0 = 0$ ,  $0.05F_0 - 147.2 (0.1698) = 0$ ,  $F_0 = 500 \text{ N}$   $F_0 = 2k\delta$ , where  $\delta = initial \text{ spring stretch}$ in equil. position.  $\delta = \frac{500}{2 \times 2.6 \times 10^3} = 0.0961 \text{ m}$   $U' = \Delta T + \Delta V_e + \Delta V_g \text{ where } U' = 0$   $\Delta T = \frac{1}{2} I_o \omega^2 - 0 = \frac{1}{2} (\frac{1}{2} \times 15 \times 0.4^2) \omega^2 = 0.6 \omega^2$   $T = \frac{1}{2} I_o \omega^2 - 0 = \frac{1}{2} (\frac{1}{2} \times 15 \times 0.4^2) \omega^2 = 0.6 \omega^2$   $T = \frac{1}{2} I_o \omega^2 - 0 = \frac{1}{2} (\frac{1}{2} \times 15 \times 0.4^2) \omega^2 = 0.6 \omega^2$   $T = \frac{1}{3\pi} I_o 0.1698 \text{ m}$   $T = \frac{1}{2} I_o \omega^2 - 0.1746^2 = -55.3 \text{ J}$ 

6/148 Each spring stretches 4 ft. So  $\Delta V_e = 2(\frac{1}{2}kx^2) = 2(\frac{1}{2}50[4]^2) = 800 \text{ ft-16}$   $\Delta V_g = -200(9-4) = -1000 \text{ ft-16}$   $U = \Delta T + \Delta V_g + \Delta V_e : 0 = \frac{1}{2} \frac{200}{32.2} v^2 - 1000 + 800$  $v^2 = 64.4, v = 8.02 \text{ ft/sec}$ 



Mass center drops 
$$h = 2(\frac{1}{2}\cos\theta) + (3\pi l)\sin\theta$$
  
=  $l(\cos\theta + 3\pi \sin\theta)$ 

$$U' = \Delta T + \Delta Vg : U' = 0$$

$$T = \frac{1}{2} I_c \omega^2 = \frac{1}{2} \left(\frac{1}{3} m l^2\right) \left(\frac{v}{l}\right)^2 = \frac{1}{6} m v^2$$

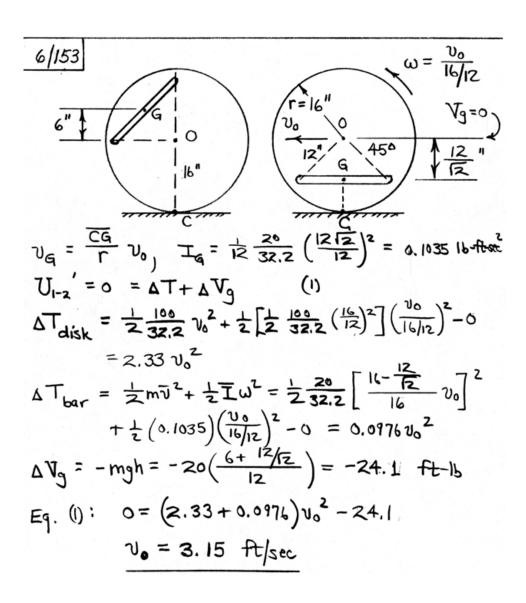
$$\Delta V_g = -mgh = -mgl \left(\cos \theta + 3\pi \sin \theta\right)$$

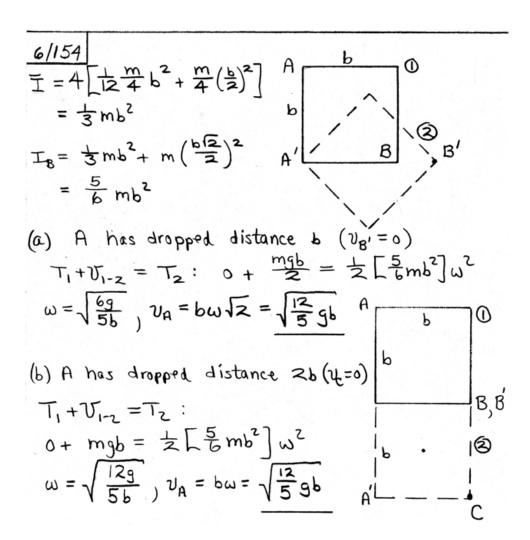
$$S_0 \quad O = \frac{1}{6} m v^2 - mgl \left(\cos \theta + 3\pi \sin \theta\right)$$

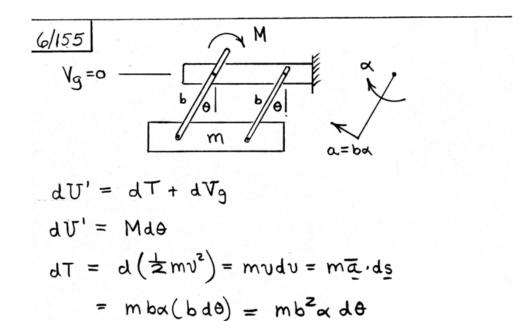
$$V = \sqrt{6gl \left(\cos \theta + 3\pi \sin \theta\right)}$$

Let p = mass of paper per unit length w = v/rFor general position m = p(L-x)  $\Delta T = \frac{1}{2} \overline{I} \omega^2 + \frac{1}{2} m \overline{v}^2$   $= \frac{1}{2} \frac{1}{2} m r^2 \omega^2 + \frac{1}{2} m v^2$   $= \frac{3}{4} m v^2 = \frac{3}{4} p(L-x) v^2$   $\Delta V_g = -pg(L-x) x \sin \theta - pg x \frac{1}{2} \sin \theta$   $= -pg x (L - \frac{x}{2}) \sin \theta$   $U = 0 = \Delta T + \Delta V_g$ ;  $0 = \frac{3}{4} p(L-x) v^2 - pg x (L - \frac{x}{2}) \sin \theta$   $v^2 = \frac{4}{3} \frac{9 x (L - x/2) \sin \theta}{L - x}$ ,  $v = 2 \sqrt{\frac{9x}{3}} \frac{L - \frac{x}{2}}{L - x} \sin \theta$ As  $x \to L$ ,  $v \to \infty$  so that the loss of potential energy  $-pg L \sin \theta/2$  is concentrated in the kinetic energy of the last bit of moving paper. Abrupt termination of motion causes abrupt energy loss at the end.

6/151 Let x = distance moved by center 0 in m.  $\theta = \tan^{-1} \frac{1}{5} = 11.31^{\circ}$ ,  $\sin \theta = 0.1961$   $\Delta V_g = mg \Delta h = mgx \sin \theta = 10(9.81)x(0.1961) = 19.24x$   $\Delta V_e = \frac{1}{2}k(x_2^2-x_1^2) = \frac{1}{2}(600)[(0.225 - \frac{275}{200}x)^2 - (0.225)^2]$   $567.2x^2 - 185.6x$   $\Delta T = \frac{1}{2}mv^2 + \frac{1}{2}I\omega^2 = \frac{1}{2}(10)v^2 + \frac{1}{2}(10)(0.125)^2(\frac{v}{0.2})^2$   $= 6.95v^2$ . For system,  $U = \Delta T + \Delta V_g + \Delta V_e$ :  $0 = 6.95v^2 + 19.24x + 567.2x^2 - 185.6x$   $v^2 = 23.93x - 81.57x^2$ Set  $\frac{dv^2}{dx} = 0$  to get x = 0.1467 m for  $v_{max}$   $v_{max}^2 = 23.93(0.1467) - 81.57(0.1467)^2$ ,  $v_{max}^2 = 1.325\frac{m}{s}$ 







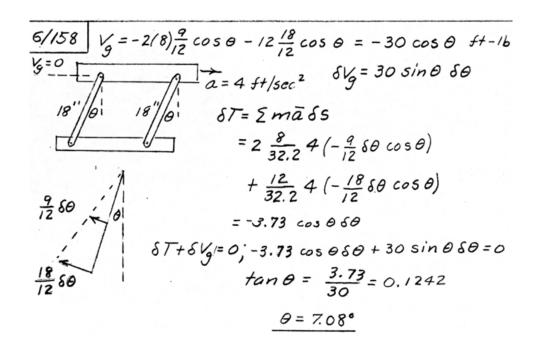
$$dV_g = d(-mgbcos\theta) = mgbsin \theta d\theta$$
Thus  $Md\theta = mb^2 \propto d\theta + mgbsin \theta d\theta$ 

$$\propto = \frac{M}{mb^2} - \frac{9}{b} sin \theta$$

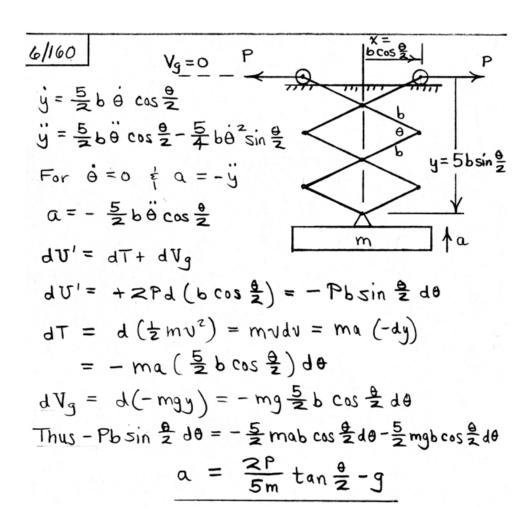
G/156 Active forces:  $I_c = \frac{1}{12}ml^2 + m\left(\frac{l}{2}\right)^2 = \frac{1}{3}ml^2$  dU' = dT + dVDue to the equilibrium condition,

The work due to the weight mg'  $I_{12}$   $I_{2}$   $I_{3}$   $I_{2}$   $I_{3}$   $I_{4}$   $I_{5}$   $I_{5}$  I

6/157 dU = dT C = instantaneous center of E = instanta

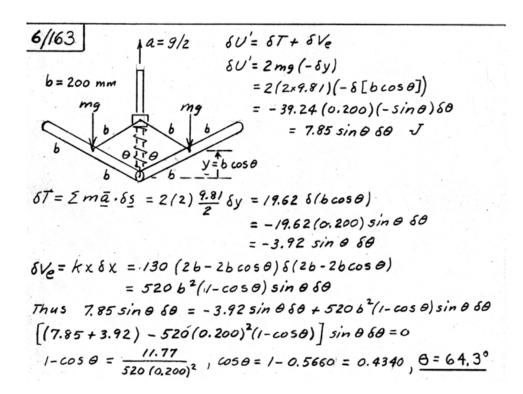


 $\frac{6/159}{dV_g = 2(6)d(8\cos\theta) + 10d(18\cos\theta)}$   $= 276 d(\cos\theta) = -276 \sin\theta d\theta \sin -16$   $dT_{bar} = d(\frac{1}{2}mv^2) = mvdv = ma_t ds$   $= \frac{10}{32.2 \times 12} (18\alpha) / 8 d\theta = 8.39 \alpha d\theta$   $= 2d(\frac{1}{2}I_0\omega^2) = 2I_0\omega d\omega = 2I_0\alpha d\theta$   $= 2(\frac{6}{32.2 \times 12} 10^2) \alpha d\theta = 3.11 \alpha d\theta$   $dT_{links} = 2d(\frac{1}{2}I_k\omega^2) = 2I_0\omega d\omega = 3.11 \alpha d\theta$   $dT_{links} = 2(18)\cos 30^\circ = 31.2 \sin 3 \cot \alpha = 2(18)\cos \frac{\theta}{2} - 18,$   $dx = 36(-\sin \frac{\theta}{2} \frac{d\theta}{2})$   $dV_e = d(\frac{1}{2}kx^2) = kx dx = -\frac{15}{12} / 8(2\cos \frac{\theta}{2} - 1) / 36\sin \frac{\theta}{2} \frac{d\theta}{2}$   $= -148.2 d\theta \sin -16$   $Thus 0 = (8.39 + 3.11) \alpha d\theta - 148.2 d\theta - 276 \sin 60^\circ d\theta$   $\alpha = 33.7 \ rad/sec^2$ 



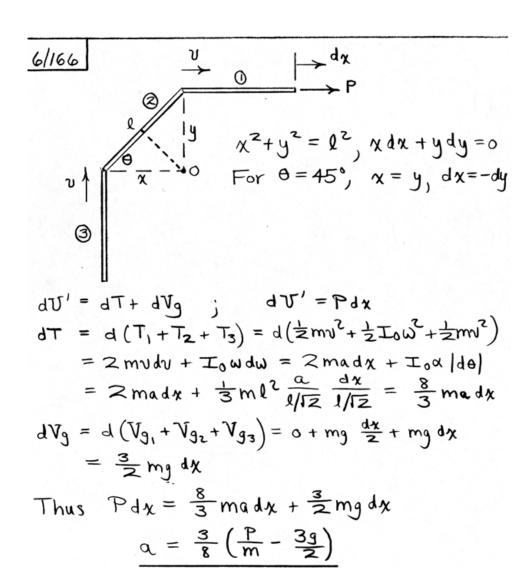
5/161 dU'=dT+dVg;  $dU'=Md\theta$   $dT=madh=mad(2bsin\theta)$   $=2mbacos\theta d\theta$   $dVg=mgdh=2mbgcos\theta d\theta$   $dVg=mgdh=2mbcos\theta(a+g)d\theta$   $a+g=\frac{M}{2mbcos\theta}$ But  $2bsin\theta=h$   $so <math>cos\theta=\sqrt{4b^2-h^2/2b}$  so  $a=\frac{M}{2mb\sqrt{1-(h/2b)^2}}-g$   $=\sqrt{1-(h/2b)^2}$ 

For a virtual displacement  $M = K\theta$   $M = K\theta$   $\delta\theta$  from the steadystate configuration,  $\delta U = \delta T$   $\delta U = -M \delta\theta + mg \delta (\bar{r} \cos \theta)$   $\delta U = -M \delta\theta - mg \bar{r} \sin \theta \delta\theta$   $\delta T = mg \cdot \delta S = ma (-\bar{r} \delta\theta \cos \theta)$   $\delta U = \delta T \cos \theta \cos \theta$   $\delta U = \delta T$ 



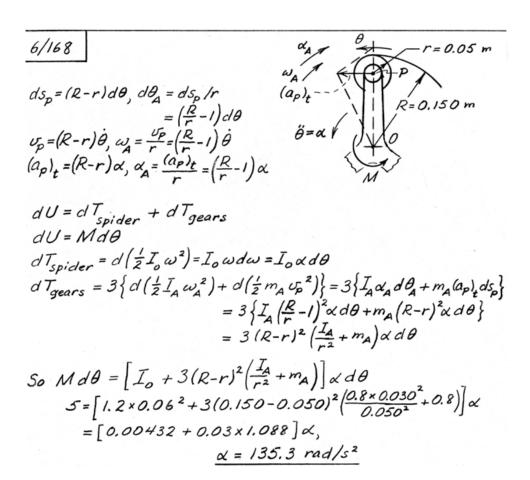
6/164 Replace P by force Pat B and couple M=Pb dU = dTdU = Pcos Ad(2bsinA) + PbdA  $= Pb(2\cos^2\theta + 1)d\theta$  $dT_{AC} = d\left(\frac{1}{2}2m\sigma^2 + \frac{1}{2}\bar{I}\omega^2\right)$  $=2m\sigma d\sigma + \bar{I}\omega d\omega = 2m\alpha dx + \bar{I}\alpha d\theta$ where  $x = 2b sin\theta$ ,  $\sigma = 2b\theta cos\theta$ ,  $a = 2b(\theta cos\theta - \theta^2 sin\theta)$ = 2h g cos & since &=0 So  $dT_{AC} = 2m(2b\ddot{\theta}\cos\theta)d(2b\sin\theta) + \frac{1}{12}(2m)(2b)^2\ddot{\theta}d\theta$  $=2mb^2(4\cos^2\theta+\frac{1}{3})\ddot{\theta}d\theta$  $dT_{oc} = d(\frac{1}{2}I_o\omega^2) = I_o\omega d\omega = I_o\omega d\theta = \frac{1}{3}mb^2\theta d\theta$ So dT=2mb2(4cos20+3) 0d0+3mb20d0 = mb2 (8 cos20+1) + de  $Pb(2\cos^2\theta+1)d\theta=mb^2(8\cos^2\theta+1)\ddot{\theta}d\theta$  $\ddot{\theta} = \alpha = \frac{P(2\cos^2\theta + 1)}{mb(8\cos^2\theta + 1)}$ 

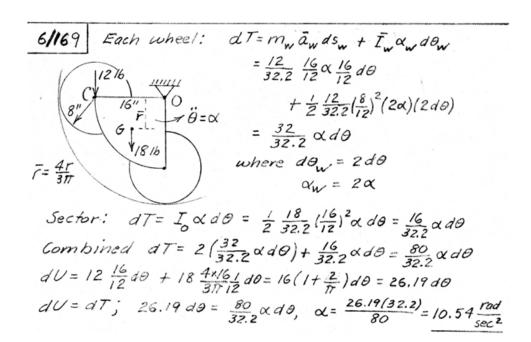
 $V_{g} = 0$   $V_{g$ 



6/167 Radius to each weight is  $r = 0.25 + 1.5 \sin \theta$  in.  $\delta T = 2 (mr\omega^2)(-\delta r) = 2 \frac{12}{16(32.2)} \frac{0.25 + 1.5 \sin \theta}{12} \omega^2(-\delta r)$  ft-16

But  $\delta r = 1.5 \cos \theta \delta \theta$  in.  $\delta T = 2(1.5) - 2(1.5) \cos \theta = 0.625 \sin \beta$ ,  $\beta = 15^\circ$ So  $\cos \theta = \frac{3 - 0.625 \sin 15^\circ}{3 - 0.625 \sin 15^\circ} = 0.9461$ ,  $\theta = 18.90^\circ$ So  $\delta T = \frac{0.25 + 1.5 \sin 18.90^\circ}{8(32.2)} \omega^2(-\frac{1.5 \cos 18.90^\circ}{12}) \delta \theta$   $\delta V_e = K \times \delta x = 5(12) \frac{2(1.5)}{12}(1 - \cos \theta) \delta \left\{ \frac{2(1.5)}{12}(1 - \cos \theta) \right\}$   $\delta U = \delta T + \delta V_e = 0$ ;  $-0.3378(10^{-3}) \omega^2 \delta \theta + 65.50(10^{-3}) \delta \theta = 0$   $\delta U = \delta T + \delta V_e = 0$ ;  $-0.3378(10^{-3}) \omega^2 \delta \theta + 65.50(10^{-3}) \delta \theta = 0$   $\delta U = \frac{65.50}{0.3378} = 193.9 \quad (rad/sec)^2$   $\omega = 13.92 \quad rad/sec, N = \frac{13.92}{2T} = 133.0 \frac{rev}{min}$ 





6/170

A

A

A

Pdy-mg 2 dy + M<sub>B</sub> d0 = d( $\frac{1}{2}mv^2$ )

y= l sin 0, dy = l cos 0 d0

mg

d( $\frac{1}{2}ml^2$ ) = ma d(2y)

= 2ma l cos 0 d0

So (P-2mg) l cos 0 + M<sub>B</sub> = 2ma l cos 0

B

MB

But P-mg = ma so

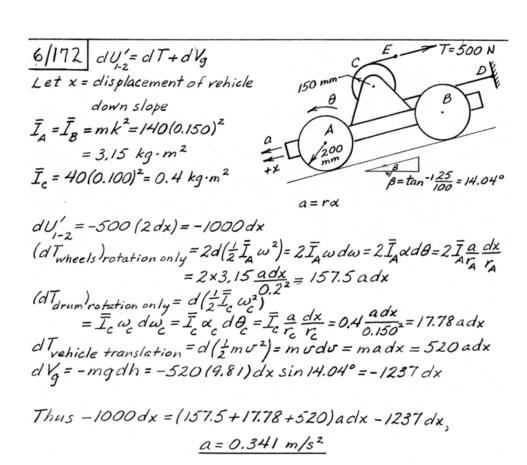
MB = mg l cos 0 ( $\frac{a}{9}$  + 1)

MB = 200(9.81)(6)(0.866)( $\frac{1.2}{9.81}$  + 1)

CAR = 1/440 N·m or 1/.44 kN·m

Lower arm;  $\Sigma M = 0$ ;  $M + M_B - Pl \cos \theta = 0$   $M = -mgl \cos \theta \left(\frac{\alpha}{g} + i\right) + mg\left(\frac{\alpha}{g} + i\right) l \cos \theta$ , M = 0M = 0 can be obtained by inspection since m is directly above C. Also, problem can be solved directly by F - m - a equations.

6/171 dV' = dT + dVq  $dV' = \sum m_i q_i \cdot ds_i + \sum \prod_i \alpha_i \cdot d\theta_i$  (  $o \cdot \frac{0.2}{m}$ + Emigdhi Let  $\begin{cases} x = \text{angular acceleration of } OA \end{cases}$   $\begin{cases} d\theta = \text{angular displacement of } OA \end{cases}$ Arm oA:  $\overline{\alpha} = \frac{0.3}{2} \propto d\overline{s} = \frac{0.3}{2} d\theta d\theta dh = -\frac{0.3}{2} d\theta$  $d\mathcal{J}'_{arm} = 4\left(\frac{0.3}{2}\right)\left(\frac{0.3}{2}d\theta\right) + 0.03 \times d\theta - 4(9.8)\left(\frac{0.3}{2}d\theta\right)$ = 0.12 × d8 - 5.89 dA Gear D: a=aA = 0.3x, d== 0.3d+, dh=-0.3d+ 00 = 30 , db0 = 3 d0  $T = mk^2 = 5(0.064)^2 = 0.0205 \text{ kg·m}^2$  $dU_{t}' = 5(0.3 \, a)(0.3 \, d\theta) + 0.0205(3 \, a)(3 \, d\theta)$  $-5(9.81)(0.340) = 0.634 \times 40 - 14.7240$ For system: dV' = dVorm + dV' = 0 0.12x d0 - 5.89 d0 + 0.634x d0 - 14.72d0 = 0  $\alpha = 27.3 \text{ rod/s}^2$ 

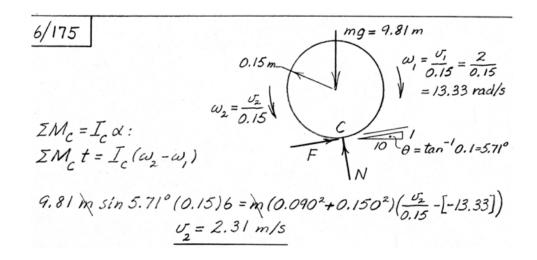


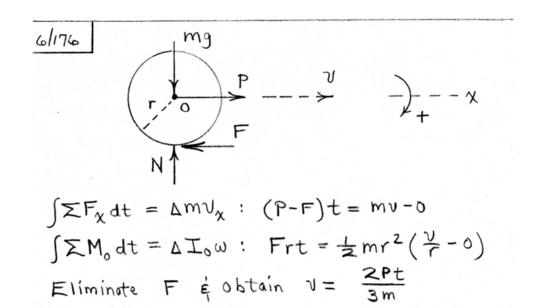
$$\int_{t_{1}}^{t_{2}} M_{0} dt = H_{02} - H_{01}$$

$$\int_{0}^{3} 90 \cos 15^{\circ} (0.8) dt = 4(\frac{1}{3})(60)(1.2)^{2} \omega$$

$$\omega = 1.811 \text{ rad/s}$$

G = my  $\overline{H}_{a}$  G = my  $\overline{H}_{a}$   $\overline{$ 

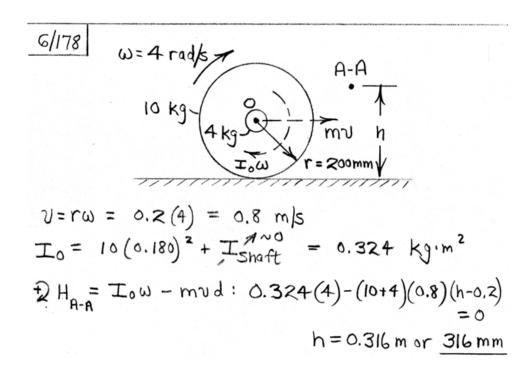




$$\int_{t_{1}}^{t_{2}} M_{q} dt = \overline{I} (\omega_{2} - \omega_{1}^{2}) = m \overline{k}^{2} \omega$$

$$\int_{0}^{3} 10 (1 - e^{-t}) dt = 75 (0.5)^{2} \omega$$

$$10 \left[ t + e^{-t} \right]_{0}^{3} = 75 (0.5)^{2} \omega, \quad \underline{\omega} = 1.093 \text{ rad/s}$$



6/179 O (Sun center)
$$-\frac{1}{d} = \frac{149.6 (109) \text{ m}}{149.6 (109) \text{ m}} = \frac{1}{149.6 (109)} = \frac{1}{149.6 ($$

System 
$$\int_{0}^{10} \sum F dt = \Delta G : 400(10) = (1200 + 800)[\nu - (-1.5)]$$
  
 $v = 0.5 \text{ m/s} \text{ (right)}$ 

Drum 
$$\int \sum M_0 dt = \Delta H_0 : 400(0.500)(10) = 800(0.480)^2 [\omega - (-3)]$$
  
 $\omega = 7.85 \text{ rad/s} CW$ 

The rotation of the drum does not affect the linear momentum of the system, so V=0.5 m/s is independent of  $\omega$ .

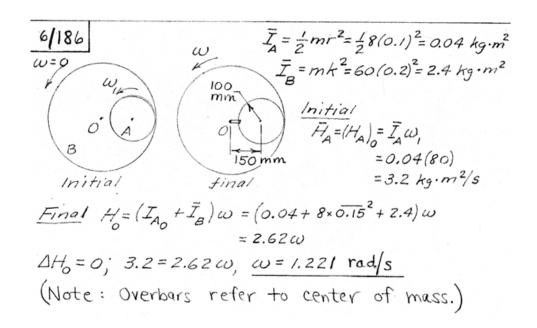
6/181  $Mdt = d(I\omega) = Id\omega$   $M = -k\omega^2 so -k\omega^2 dt = Id\omega$   $-k \int_0^t dt = I \int_0^t \frac{d\omega}{\omega^2} dt - kt = I(-\frac{1}{\omega})_{\omega_0}^{\omega_0/2} = I(\frac{-1}{\omega_0/2} + \frac{1}{\omega_0})$   $= -I/\omega_0$   $so t = \frac{I}{\omega_0 k}$ 

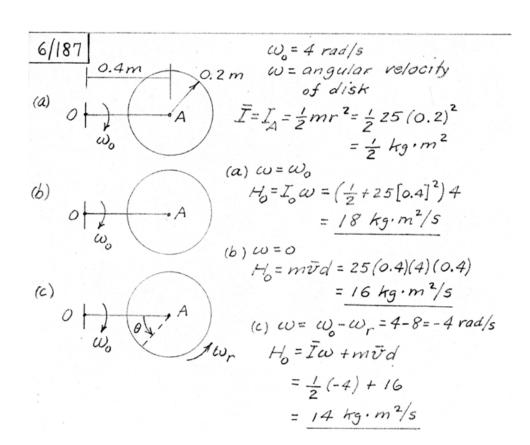
6/182 For no slipping 4 mass center at 0,  $ZM_c = I_c \propto so \int ZM_c dt = \Delta H_c = \Delta (I_c \omega)$   $I_c = m(k_o^2 + r^2) = 2(0.060^2 + 0.080^2)$   $= 0.02 kg \cdot m^2$   $= 0.02 kg \cdot m^2$   $= 0.02 \left(\frac{\sigma}{0.080} - \left[\frac{-0.3}{0.080}\right]\right)$   $0.469 = 0.25 (\sigma + 0.3)$   $0.469 = 0.25 (\sigma + 0.3)$   $0.80F(5) - 0.3(5) = 0.09(\sigma + 0$ 

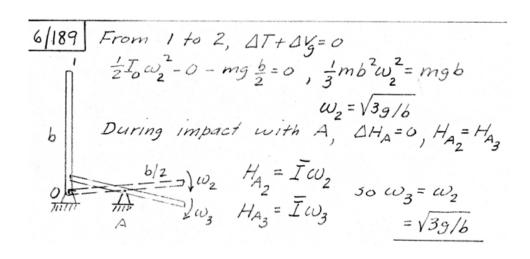
6/183  $H_{0_1} = H_{0_2}$ :  $mvb = (I_0 + mb^2) \omega$   $\frac{2}{16} \frac{1}{32.2} (1500) \frac{15}{12} = \left[\frac{1}{3} \frac{20}{32.2} (\frac{30}{12})^2 + \frac{2}{16} \frac{1}{32.2} (\frac{15}{12})^2\right] \omega$  $\omega = 5.60 \text{ rad/sec}$  6/184  $\int \sum M_{G} dt = \overline{H}_{2} - \overline{H}_{1}$ :  $O_{\chi} \frac{15}{12} (0.001) = \frac{1}{12} \frac{20}{32.2} (\frac{30}{12})^{2} (\omega - 0)$ Where  $\omega = 5.60 \text{ rad/sec from}$ Frob. 6/183.  $\omega$   $0_{\chi} = 1449 \text{ lb}$ 

6/185 f  $H_{01} = H_{02}$  for system

muh =  $(I_0 + mh^2) \omega$   $\left(\frac{1/16}{32.2}\right) \left(\frac{43}{12}\right) = \left[\frac{55}{32.2}\left(\frac{37}{12}\right)^2 + \frac{1/16}{32.2}\left(\frac{43}{12}\right)^2\right] \omega$   $\omega = 0.684 \text{ rad/sec}$ 

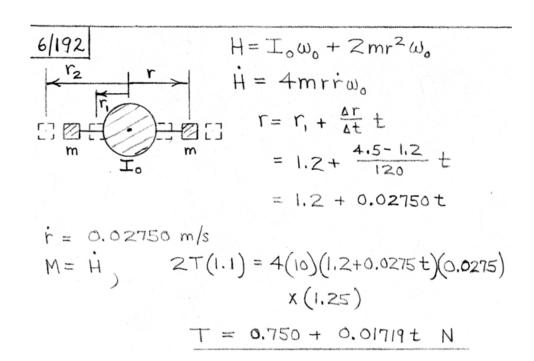


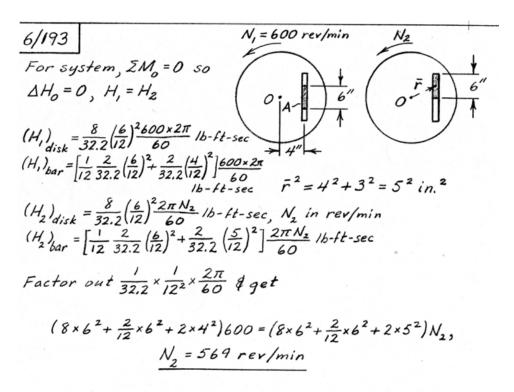




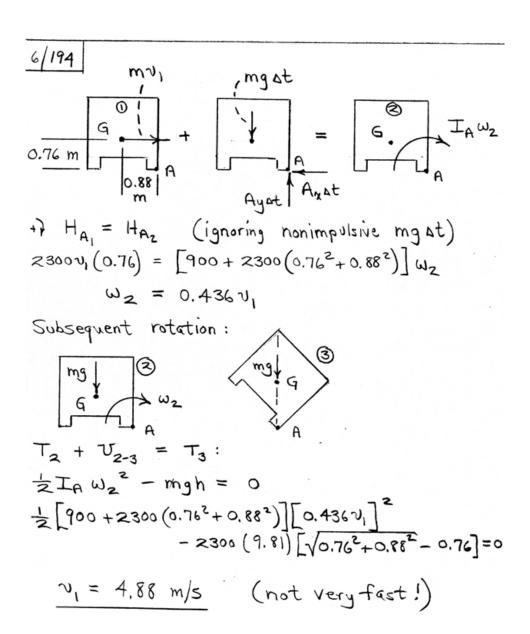
6/190 Approximate the diver's body as a uniform slender box in the first case and as a sphere in the second case. Conservation of angular momentum  $H_1 = H_2$ :  $\frac{1}{12} M_1^2 N_1 = \frac{2}{5} M_1^2 N_2$   $\frac{1}{12} (2)^2 (0.3) = \frac{2}{5} (\frac{0.7}{2})^2 N_2$   $N_2 = 2.04 \text{ rev/s}$ 

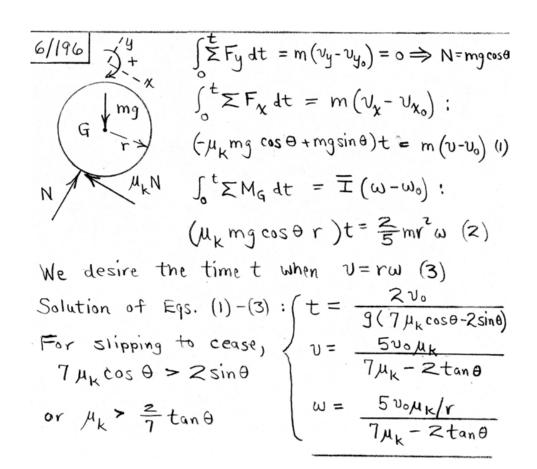
 $\frac{6/191}{Initial conditions:} \sum M_o = 0 = \Delta H_o so H_{o_1} = H_{o_2}$ 0A = 110 mm AG=80mm (I) each red = 0.84(12 × 0.160 + 0.1362) = 0.01733 kg·m2  $(I_o)_{disk} = 30(0.090)^2 = 0.243 \text{ kg·m}^2$  $\omega_1 = 600 \times 2\pi/60 = 62.8 \text{ rad/s}$  $H_{0} = [4(0.01733) + 0.243]62.8 = 19.62 \text{ kg} \cdot \text{m}^2/\text{s}$ Final conditions:  $(I_o)_{each\ rod} = 0.84(\frac{1}{12} \times 0.160^2 + [0.110 + 0.080]^2) = 0.0321 \text{ kg·m}^2$ (Io) disk = 0.243 kg·m2  $H_0 = [4(0.0321) + 0.243] \omega_2 = 0.371 \omega_2$ Thus 19.62 = 0.371 w2, w2 = 52.8 rad/s, N=504 rev/min Energy loss:  $T = \sum_{i=1}^{1} I_{i} \omega^{2} = \frac{1}{2} (4 \times 0.01733 + 0.243)(62.8)^{2} = 617 J$  $T_2 = \sum_{i=1}^{4} I_0' \omega'^2 = \frac{1}{2} (4 \times 0.0321 + 0.243)(52.8)^2 = 518$  J AE = T, -T, = 617-518 = 98.1 J loss Direction of rotation & sequence of rod release do not affect the results.





Friction forces in the slot are internal so have no effect on  $ZM_0$ . Hence the final value of  $N_2$ , as well as the loss of energy, is unaffected.



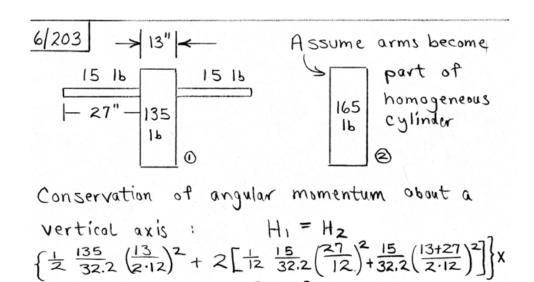


6/198 Conservation of angular momentum about the vertical spin axis of the platform:  $H_1 = H_2$   $\left[10(0.3)^2\right](250 \frac{2\pi}{60}) = \left[1 + \frac{1}{2}(10)(0.3)^2 + 10(0.6)^2\right] \times (30 \frac{2\pi}{60})$   $I = 3.45 \text{ kg} \cdot \text{m}^2$ 

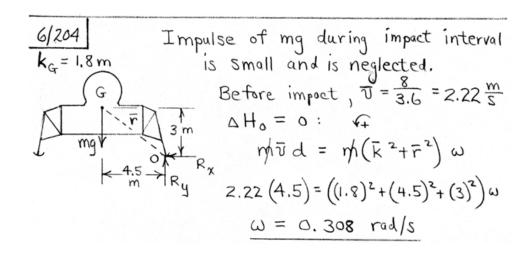
6/199 Conservation of angular momentum about the vertical spin axis of the platform:  $H_1 = H_2$   $[10(0.3)^2][250] = [3.45 + 10(0.6)^2]N$   $-10(0.3)^2[250]$  N = 63.8 rev/min

6/200 Bar B:  $U_{1-2} = 0 = \Delta T + \Delta V_g$   $\Delta V_g = -mgh = -8(9.81)(0.180) = -14.13 J$   $\Delta T = \frac{1}{2}I\omega_B^2 = \frac{1}{2}(8)(0.220)^2\omega_B^2 = 0.1936\omega_B^2$ So  $0 = 0.1936\omega_B^2 - 14.13$ ,  $\omega_B = 8.54 \text{ rod/s}$ Prior to impact:  $H_0 = I\omega_B = 8(0.220)^2(8.54) = 3.31\frac{kg \cdot m^2}{s}$ For system after import:  $H_0 = I_{tot}\omega = [z \cdot zo(0.3)^2 + 8(0.220)^2] \omega = 3.99\omega$   $\Delta H_0 = 0: 3.99\omega - 3.31 = 0, \quad \omega = 0.830 \text{ rad/s}$ After import:  $U_{1-2}' = 0 = \Delta T + \Delta V_g$   $\Delta V_g = mgh = 2.720(9.81)(0.25)(1-cos\theta)$   $+8(9.81)(0.18)(1-cos\theta) = 112.2(1-cos\theta)$   $\Delta T = 0 - \frac{1}{2}I\omega^2 = \frac{1}{2}[z \cdot 2o(0.3)^2 + 8(0.220)^2](0.830)^2$  = -1.372 JSo  $0 = 112.2(1-cos\theta) - 1.372$ ,  $\Theta = 8.97^{\circ}$   $Loss of energy <math>|\Delta E| = (V_g)_{before} - (V_g)_{after}$  = 14.13 - 112.2(1-cos8.97') = 12.75J

6/201  $\Delta H = 0$ ; Initial:  $H_{rods} = 2I\omega = 2(1.5)(0.060)^2 \frac{300 \times 2\pi}{60} \text{ N·m·s}$   $H_{base} = mk^2\omega = 4(0.040)^2 \frac{300 \times 2\pi}{60} \text{ N·m·s}$  Final:  $H_{rods} = 2[I + md^2]\omega = 2m[\frac{l^2}{12} + d^2]\frac{2\pi N}{60}$   $= 2(1.5)[\frac{0.3^2}{12} + (0.150 + 0.060)^2]\frac{2\pi N}{60}$   $= 0.1548(\frac{2\pi N}{60}) \text{ N·m·s}$   $H_{base} = 4(0.040)^2 \frac{2\pi N}{60} = 0.0064(\frac{2\pi N}{60})$ Thus  $[3(0.06)^2 + 4(0.04)^2]300 = [0.1548 + 0.0064]N$ 0.0172(300) = 0.1612 N, N = 32.0 rev/min Neglecting impulse of weight,  $\Delta H_A = 0$   $\frac{1}{45^{\circ}}$  during impact:  $m_1 = \frac{1}{2} \sin \alpha = \frac{1}{3} m \ell^2 \omega_2$   $\omega_2 = \frac{3v_1}{2\ell} \sin \alpha$ Ax Ay During subsequent rotation about A,  $v = \Delta T$  or  $-mg = \frac{1}{2} \left(1 - \cos 45^{\circ}\right) = 0 - \frac{1}{2} I_A \omega_2^2$   $\omega_2 = \sqrt{\frac{39}{2} \left(1 - \frac{\sqrt{2}}{2}\right)}$ So  $\sqrt{\frac{39}{2} \left(1 - \frac{\sqrt{2}}{2}\right)} = \frac{3v_1}{2\ell} \sin \alpha$ Sin  $\alpha = \frac{0.625}{v_1} \sqrt{g\ell}$   $\left(0 \le \alpha \le 45^{\circ}\right)$ 



 $1 = \left\{ \frac{1}{2} \frac{165}{32.2} \left( \frac{13}{2.12} \right)^2 \right\} N$  N = 4.78 rev/sec



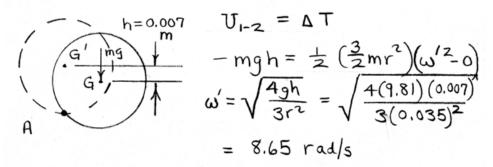
Velocity of bar at impact =  $\sqrt{2gh} = \overline{v}$ Alz Velocity of bar at impact =  $\sqrt{2gh} = \overline{v}$ Neglect Small impulse of weight.

Algorithms  $\Delta H_B = 0$ Thus  $\Delta H_B = 0$ Thu

Angular impulse of mg is negligible.

Before impact:  $H_A = \overline{L}\omega + mv(r-h)$   $= mk^2 \frac{v}{r} + mv(r-h)$   $= mk^2 \frac{v}{r} + mv(r-h)$ Just after impact:  $H_A' = \overline{L_A} \frac{v'}{r} = m(k^2 + r^2) \frac{v'}{r}$   $\Delta H_A = 0: mv(\frac{k^2}{r} + r - h) = m(k^2 + r^2) \frac{v'}{r}$   $v' = v(1 - \frac{rh}{k^2 + r^2})$ During roll on curb point,  $\Delta T + \Delta V_g = 0$   $\left[0 - \frac{1}{2}m(k^2 + r^2) \frac{v'^2}{r^2}\right] + \left[mgh - 0\right] = 0$ Solve for  $v: v = \frac{r}{k^2 + r^2 - rh} \sqrt{2gh(k^2 + r^2)}$ 

6/207 Process II - roll about fixed point A



Process I - impact at A

$$\Delta H_{A} = 0: mv(r-h) = I_{A}\omega'$$

$$= (\frac{3}{2}mr^{2})\omega'$$

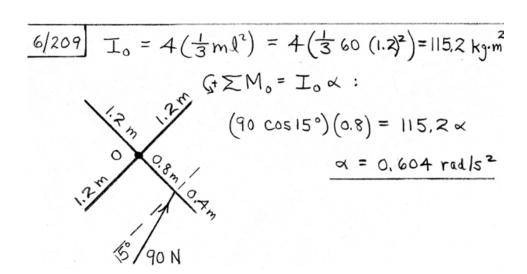
$$\Delta H_{A} = 0: mv(r-h) = I_{A}\omega'$$

$$= (\frac{3}{2}mr^{2})\omega'$$

$$\Delta = \frac{3r^{2}\omega'}{r-h} = \frac{3(0.035)^{2}(8.65)}{0.035 - 0.007} = 1.135 \frac{rad}{5}$$

 $\begin{array}{lll} \blacktriangleright 6/208 & During \ slipping \ (a_o)_x = 0, \ so \\ \hline \Sigma F_x = 0, \ F - mg \sin\theta = 0, \ F = \mu_k mg \cos\theta \\ so \ mg \sin\theta = \mu_k mg \cos\theta, \\ \mu_k = \tan\theta = \tan 10^{\circ} \\ \mu_k = 0.1763 \\ \hline \Sigma M_o \times t = \Delta H_o: \\ 0.1763 \ (30) \ (9.81) \cos 10^{\circ} \ (0.1) t \\ &= 0 - (-30 \times 0.075^2) \frac{2\pi \times 300}{60}, \ t = 1.037s \\ \hline During \ rolling \ (assume \ no \ slip) \\ \int \Sigma F_x \ dt = m\Delta v_x: \ (30 \times 9.81 \sin 10^{\circ} - F) \ 4 = 30 \ (v - 0), \ 204 - 4F = 30v \\ \int M_o \ dt = I_o \Delta w: \ 0.1F \times 4 = 30 \times 0.075^2 \ (v/0.1), \ 4F = 16.88 \ v \\ Combine \ get \ F = 18.40 \ N, \ v = 4.36 \ m/s \end{array}$ 

Check:  $F_{max} = \mu_s N_s \not= \mu_k N = 0.1763 \times 30 \times 9.81 \cos 10^\circ = 51.1 N < \mu_s N$ so  $18.40 < \mu_k N < \mu_s N \not= assumption of no slip is valid.$ 

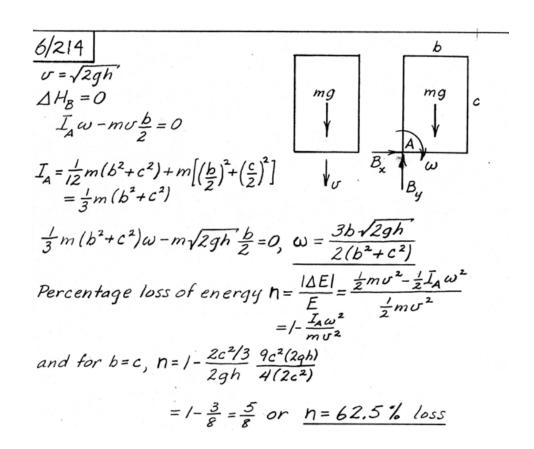


 $\frac{6|210}{\omega = \frac{1}{2}m} = \frac{1}{2}mr^{2} \left(\frac{7}{r}\right) = \frac{1$ 

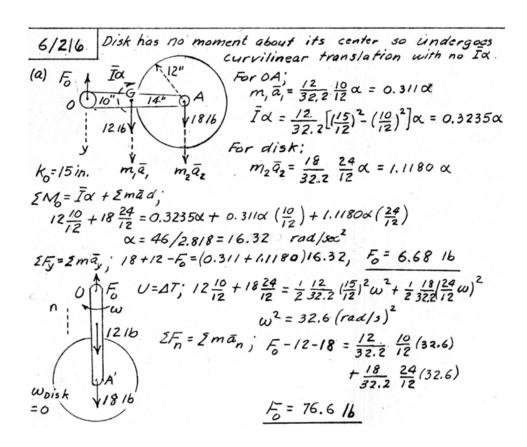
 $\begin{array}{l} G/211 \\ F_{+} \sum M_{0} = I_{0} \ddot{\theta} : \\ -mg \stackrel{?}{>} \sin \theta = \frac{1}{3} m \ell^{2} \ddot{\theta} \\ \ddot{\theta} = -\frac{3g}{2\ell} \sin \theta \\ \dot{\theta} d\dot{\theta} = \ddot{\theta} d\theta \\ \Rightarrow \omega^{2} = \omega_{0}^{2} - \frac{3g}{2\ell} (1 - \cos \theta) \\ When <math>\theta = \beta_{1} \omega = 0 : 0 = \omega_{0}^{2} - \frac{3g}{2\ell} (1 - \cos \beta) \\ \omega_{0}^{2} = \frac{3g}{2\ell} (1 - \cos \beta) \\ \Rightarrow \frac{d\theta}{dt} = \sqrt{\frac{3g}{2\ell}} \sqrt{\cos \theta - \cos \beta} \\ \Rightarrow \frac{d\theta}{dt} = \sqrt{\frac{3g}{2\ell}} \sqrt{\cos \theta - \cos \beta} \\ S_{0} & t = \sqrt{\frac{3g}{3g}} \int_{0}^{\beta} \frac{d\theta}{\sqrt{\cos \theta - \cos \beta}} \\ \end{array}$ 

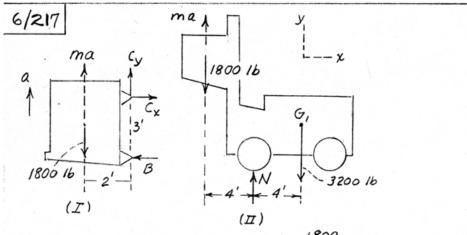
6/212 Max. power occurs when dV /dt is greatest, which occurs when  $\overline{v}_y$  is max. at the start.  $\overline{v}_y = 1.500 \, \omega = 1.500 \, \frac{4\pi}{180} = 0.1047 \, \text{m/s}$   $P = mg \, \overline{v}_y = 1600(5) \, 9.81(0.1047) = 8218 \, \text{W}$ or  $P = 8.22 \, \text{kW}$ 

6/213  $\Delta V_g + \Delta V_e + \Delta T = 0$   $\Delta V_g = -15(2) = -30 \text{ ft} - 16$   $\Delta V_e = \frac{1}{2} 3 (\sqrt{6^2 + 4^2} - 2)^2 = 40.74 \text{ ft} - 16$   $\Delta T = 0 - \frac{1}{2} \frac{1}{3} \frac{15}{32.2} A^2 \omega^2 = -1.242 \omega^2$  $-30 + 40.74 - 1.242 \omega^2 = 0, \quad \omega^2 = 8.64, \quad \omega = 2.94 \text{ rad/sec}$ 



 $I = 4 \left[ \frac{1}{12} m (2r^2) + m \left( \frac{r}{12} \right)^2 \right]$   $= \frac{8}{3} m r^2$   $\sum F_y = 0: N = 4 mg \cos \theta$   $\sum F_x = m a_{gx}: 4 mg \sin \theta - F = 4 ma$   $\sum M_q = I \propto : Fr = \frac{8}{3} m r^2 \propto$   $No slipping: a = r \propto$   $Solution of (1) - (3): \begin{cases} a = \frac{3}{5} g \sin \theta, \alpha = \frac{39}{5r} \sin \theta \end{cases}$   $F = \frac{8}{5} mg \sin \theta$   $H_s = \frac{F}{N} = \frac{\frac{8}{5} mg \sin \theta}{4 mg \cos \theta} = \frac{2}{5} \tan \theta$ 



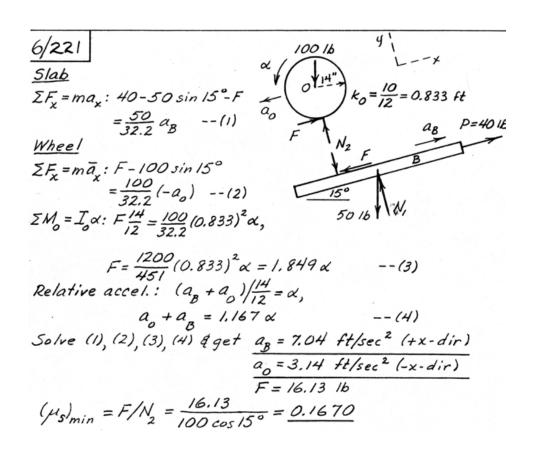


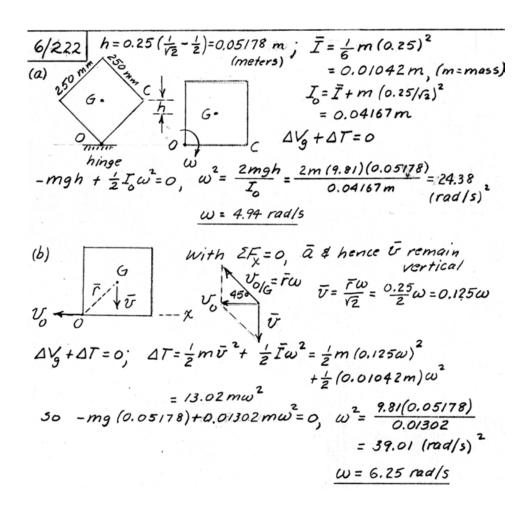
(II) 
$$\geq M_c = m\bar{\alpha}d$$
;  $3200(4) - 1800(4) = \frac{1800}{32.2}a(4)$ ,  $a = 25.04$   
 $ft/\sec^2$   
(I)  $\geq M_c = m\bar{\alpha}d$ ;  $3B - 2(1800) = \frac{(800)}{32.2}(25.04)(2)$ 

(I) 
$$\sum M_c = m\bar{a}d$$
,  $3B - 2(1800) = \frac{(800)}{32.2}(25.04)(2)$   
 $B = 2/30 \text{ lb}$ 

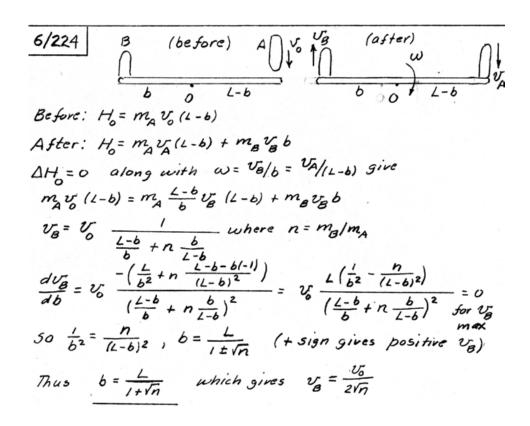
6/219 For the entire spacecraft,  $\sum M_{\chi} = I_{\chi} \propto : 10^{-6} = 150,000 \, \text{d}$   $\propto = 6.67 \times 10^{-12} \, \text{rad/s}^2$   $\Theta = \Theta_0 + \omega_0 t + \frac{1}{2} \times t^2$   $\frac{1}{3600} \left( \frac{M}{180} \right) = 0 + 0 + \frac{1}{2} \left( 6.67 \times 10^{-12} \right) t^2$   $t = 1206 \, \text{s}$ 

5/220 For system  $U = \Delta T + \Delta V_g + \Delta V_e$   $U = 0, \Delta T = \frac{1}{2}I_0\omega^2 = \frac{1}{2}(\frac{1}{3}mL^2)(\frac{V_A}{L})^2$   $= \frac{1}{6}mV_A^2 = \frac{1}{6}\frac{60}{32.2}V_A^2 = 0.3106V_A^2$   $\Delta V_e = \frac{1}{2}kx^2 - 0$   $= \frac{1}{2}10(5-1)^2 = 80 \text{ ft-16}$   $V_A \Delta V_g = -60(2) = -120 \text{ ft-16}$ Thus  $0 = 0.3106V_A^2 - 120 + 80$   $V_A^2 = 128.8, V_A = 11.35 \text{ ft/sec}$ 





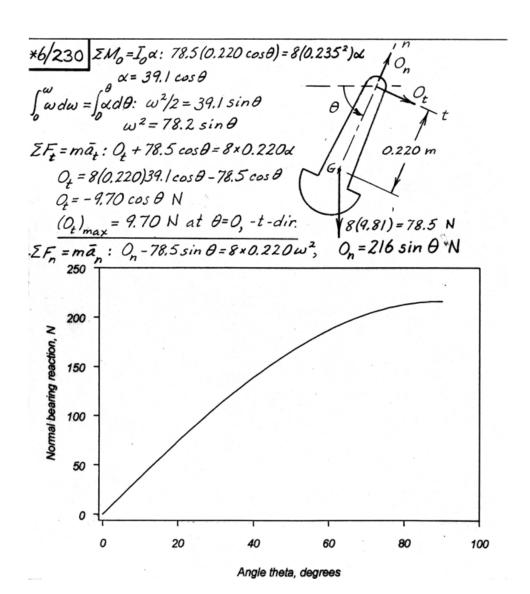
6/223  $\delta T_{both bolls} = 2ma_r \delta r$ ,  $r = \frac{1}{12}(1+6\sin\beta)$  ft  $a_r = -r\omega^2 = -r(15.71)^2$   $= -20.56(1+6\sin\beta)\frac{ft}{5\pi c^2}$   $\delta T = -2\frac{3}{32.2}(20.56)(1+6\sin\beta)\frac{cos\beta}{2}\delta\beta$   $\delta V_g = -20\delta h_2 - 2(3)\delta h_s$   $\delta V_g = -20\delta h$ 

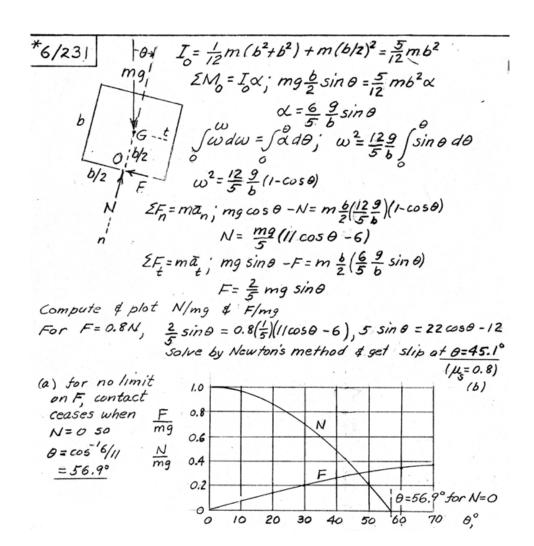


(a) Max. acceleration occurs when  $F = \mu N_A = 0.8 N_A$ (b)  $F = \mu N_A = 0.8 N_A$ (c)  $F = \mu N_A = 0.8 N_A$ (c)  $F = \mu N_A = 0.8 N_A$ (d)  $F = \mu N_A = 0.8 N_A$ (e)  $F = \mu N_A = 0.8 N_A$ (f)  $F = \mu N_A = 0.8 N_A$ (g)  $F = 0.8 (92.33) = 73.86 N_A$ (h)  $F = 0.8 (92.33) = 73.86 N_A$ (g)  $F = 0.8 (92.33) = 73.86 N_A$ (h)  $F = 0.8 (92.33) = 73.86 N_A$ (h) F

►6/228  $\bar{I} = \frac{1}{12}mt^2 = \frac{1}{12}(4)(1.2)^2 = 0.48 \text{ kg·m}^2$ , Fdt = 14 N·s  $C_y$   $C_x$   $C_y$   $C_y$  C

►6/229 Fixed-axis rotation  $\Sigma F_n = m\bar{a}_n : T - 150 = \frac{150}{32.2} \frac{13^2}{92/12},$  T = 253 1b15016  $\theta = \cos^{-1}(10/23) = 64.2^{\circ}$ T=253 16 B= 0-18°=46.2° ZF = 0: 253-Rcos18°-Pcos46.2°=0 IF, =0: Rsin 18°-Pcos 46.2°=0 Solve & get P=86.716, R=20316  $\gamma = \sin^{-1}\frac{13}{92} = 8.12^{\circ}$  $\Sigma F_{t} = m\bar{a}_{t}$ : 203 sin 18° -  $F_{t} = \frac{75}{32.2} \frac{13^{2}}{92/12} \sin 8.12$ F = 55.416 mān, k T+ ZMo = Io x = 0: 203 sin 18° (92-18.18) -75 (13) +55.4 (92) +M = 0 M = 504 lb-in. 203 lb





\*6/232 
$$U' = \Delta T + \Delta V_e + \Delta V_g$$
;  $U' = 0$ 

$$\Delta T = \frac{1}{2} m \sigma^2 - 0 = \frac{1}{2} \frac{10}{32.2} \sigma^2 ft - 16$$

$$\Delta V_e = \frac{2}{2} k (x_2^2 - x_1^2) = 6 \left[ (\sqrt{x^2 + 12^2} - 12)^2 - (15 - 12)^2 \right] \frac{1}{12}$$

$$= \frac{1}{2} \left[ x^2 - 24 \sqrt{x^2 + 144} + 279 \right] ft - 16$$

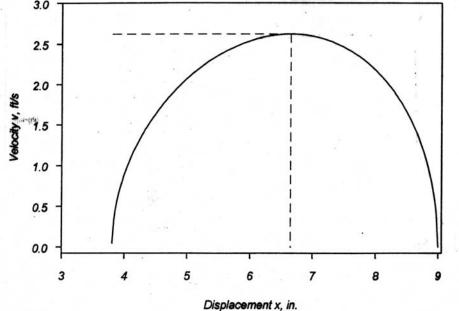
$$\frac{5}{32.2} \sigma^2 + \frac{x^2}{2} - 12 \sqrt{x^2 + 144} + \frac{279}{2} + \frac{15}{2} - \frac{5x}{6} = 0$$

$$\sigma^2 = \frac{32.2}{5} \left\{ 12 \sqrt{x^2 + 144} - \frac{x^2}{2} + \frac{5x}{6} - 147 \right\} (ft/sec)^2$$
(where x is in inches)

Plot  $\sigma$  vs. x (see continuation)
$$\sigma = 0 \text{ at } x = 3.81 \text{ in.}$$

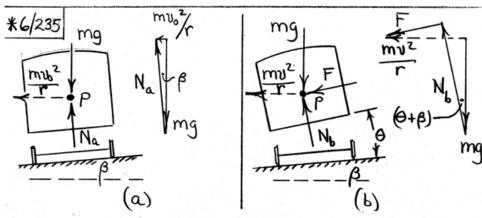
$$\sigma_{max} = 2.62 \text{ ft/sec at } x = 6.65 \text{ in.}$$
3.0

2.5



\*6/233  $U = \Delta T$ :  $T = \frac{1}{2} I_c \omega^2 = \frac{1}{2} \frac{1}{3} \frac{W}{9} 4^2 \omega^2$  $U = Wh = W(2 - 2\cos\theta)$  $=2W(1-\cos\theta)$ Thus  $2W(1-\cos\theta) = \frac{8W}{3g}\omega^2$ ,  $\omega^2 = \frac{3g}{4}(1-\cos\theta)$   $\omega = \sqrt{(3\times32.2/4)(1-\cos\theta)} = 4.91\sqrt{1-\cos\theta}$  rad/sec  $U_A = U_{A/B} \cos \theta = L \cos \theta$   $= 4 (4.91) \sqrt{1 - \cos \theta} \cos \theta \text{ ft/sec}$ UA = UB + UA/B: = 19.66 cos 0-11-cos0 = 7.57 ft/sec 6 Velocity of end A, ft/s 2 0 20 40 60 80 100 Angle theta, degrees

\*6/234 From the solution of Prob. 6/23,  $K\theta - \frac{5}{2} mgl \sin \theta - \frac{5}{2} mal \cos \theta = 0$  With numbers:  $75\theta - 7.36 \sin \theta - 14.72 \cos \theta = 0$  Numerical solution:  $\theta = 12.17^{\circ}$ 



(Passenger is shown as particle P above)
Note that F = 0.3 mv<sup>2</sup>/r

Note That 
$$F = \frac{mv_0^2/r}{mg} = \frac{v_0^2}{gr} = \frac{(160/3.6)^2}{9.81(1900)}$$
  
 $\beta = 6.05^\circ$ 

(b) From the force polygon,  

$$mg \sin (\theta + \beta) + \frac{0.3mv^2}{r^2} = \frac{mv^2}{r} \cos (\theta + \beta)$$
  
9.81  $\sin (\theta + \beta) + \frac{(260/3.6)^2}{1900} (0.3 - \cos (\theta + \beta)) = 0$   
9.81  $\sin (\theta + \beta) + 2.75 [0.3 - \cos(\theta + \beta)] = 0$ 

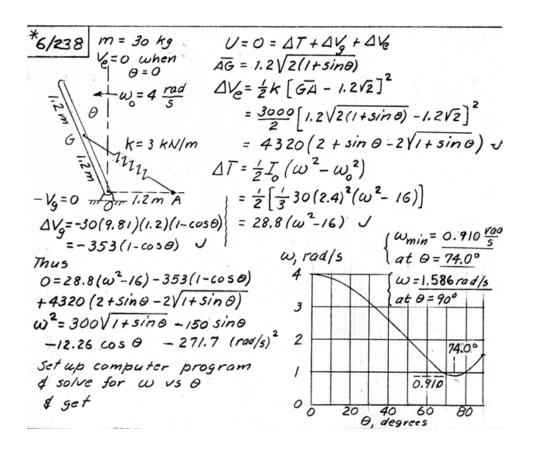
Numerical solution:  $\theta = 4.95^{\circ}$ 

G+ 
$$\sum M_A = I_A \alpha$$
: 300 (3  $\sin \beta$ ) - 981 ( $2 \sin \theta$ ) = 533 $\alpha$ 

$$\frac{900}{\sqrt{2}} \left( \sin \frac{\theta}{2} + \cos \frac{\theta}{2} \right) - 1962 \sin \theta = 533\alpha \quad (1)$$

$$\omega d\omega = \int_0^{\theta} \alpha d\theta : \omega^2 = \frac{2}{533} \int_0^{\theta} \left[ \frac{900}{\sqrt{2}} \left( \sin \frac{\theta}{2} + \cos \frac{\theta}{2} \right) - 1962 \sin \theta \right] d\theta$$
or  $\omega^2 = \frac{2}{533} \left[ \frac{1800}{\sqrt{2}} \left( 1 - \cos \frac{\theta}{2} + \sin \frac{\theta}{2} \right) - 1962 \left( 1 - \cos \theta \right) \right] (2)$ 

- (a) For max  $\omega$ , set  $\alpha = 0$  in (1) f solve for  $\theta$ :  $\theta = 22.4^{\circ} \qquad \text{From } (2) : \underline{\omega_{\text{max}}} = 0.680 \frac{\text{rad}}{\text{S}}$
- (b) Solve (2) for w = 0: 0 max = 45.9°



$$\frac{*6/239}{O_X} \qquad \overline{r} = \frac{\sum m\overline{r}}{\sum m} = \frac{2m (1/2) + m(31/4)}{3m} = \frac{7}{12} l$$

$$I_0 = \frac{1}{3} (2m) l^2 + \left[\frac{1}{2} m (\frac{l}{4})^2 + m(\frac{31}{4})^2\right]$$

$$= \frac{|Z|}{96} m l^2$$

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$$= \frac{168}{3} \frac{9}{12} \cos \theta$$

$$\Rightarrow \omega = \frac{168}{3} \frac{9}{12} \frac{1}{3} \cos \theta$$

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$$\Rightarrow \omega = \frac{168}{3} \frac{9}{12} \frac{1}{3} \frac{1}{3} \sin \theta$$

$$\Rightarrow t = \int_0^{\theta} \frac{d\theta}{\left[\omega_0^2 + \frac{336}{321} \frac{9}{3} \sin \theta\right]^{1/2}}$$
Numerical solution with 
$$\begin{cases} \omega_0 = 3 \text{ rad/s} \\ l = 0.8 \text{ m} \\ \theta = \frac{1}{12} \frac{1}{2} \sin \theta \end{cases}$$

\*6/240 (+ ZM = I = : 丒 mg & sin 0 = 3 mb2 0  $\ddot{\theta} = \frac{3}{2} \frac{9}{5} \sin \theta$  $\dot{\theta} d\dot{\theta} = \ddot{\theta} d\theta$ ;  $\int_{0}^{4} \dot{\theta} d\dot{\theta} = \frac{3q}{2b} \int_{0}^{4} \sin\theta d\theta$  $\frac{\dot{\theta}^2}{3} - \frac{\theta_0^2}{3} = \frac{39}{2h} \left( \cos \theta_0 - \cos \theta \right)$  $\frac{d\theta}{dt} = \left[\dot{\theta}_0^2 + \frac{39}{b} \left(\cos\theta_0 - \cos\theta\right)\right]^{1/2}$  $\int_{0}^{t} dt = \int_{0}^{t} \frac{d\theta}{\left[\dot{\theta}_{0}^{2} + \frac{3\theta}{b} \left(c\omega \theta_{0} - c\omega \theta\right)\right]^{1/2}}$ With  $\theta_0 = 10^{\circ} (0.1745 \text{ rad})$ , b = 60',  $g = 32.2 \frac{77}{\text{sec}^2}$ and  $\dot{\theta}_0 = \frac{(v_A)_0}{L} = \frac{4.5}{60} = 0.0750 \text{ rod/sec}, a$ numerical solution yields t= 2.85 sec Energy considerations from to = 100 to 0 = 900:  $\Delta T + \Delta V_9 = 0$   $\Delta T = \frac{1}{2} \pm 0 \left[ \frac{v_A}{b} \right]^2 - \frac{1}{2} \pm 0 \left[ \frac{(v_A)_0}{b} \right]^2 - \frac{1}{6} m \left[ v_A^2 - (v_A)_0^2 \right]$ A Vg = - mg h = - mg \( \frac{b}{2} \) cas 10° 50 to [vn2 - 4.52] - x (32.2) 2 cos 10° = 0 UA = 75.7 ft/sec